

Experimental Comparison of the Known Hypotheses of the Lateral Buckling for Semi-Slender Pinned Columns

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Abstract: The paper presents the comparison of experimental results obtained from tests on semi-slender columns with pinned ends made of steel R35 to simplifications and hypotheses of loss of stability by lateral buckling in elastic-plastic states of columns axially compressed by force. The Tetmajer-Jasiński's and Johnson-Ostenfeld's simplifications, as well as the hypotheses given by Engesser and Kármán and Shanley, Ylinen, Březina, Pearson and Bleich and Vol'mir and the author's one approximated, are analyzed. The graphs of surface functions of critical compressive stress $\sigma_{cr}(A, L^*t)$ depending on a cross-section area and length times thickness product are presented as the theoretical examples of thin-walled cylindrical and square columns made of steel R35. In order to compare the experimental results with other simplifications and hypotheses are shown in the adequate ranges for elastic-plastic states as the graphs of the functions of critical compressive stress depending on slenderness ratio $\sigma_{cr}(\lambda)$.

Keywords: Stability, Buckling, Elastic-Plastic States, Semi-Slender, Column

Introduction

When is considered the application of shell elements in load-bearing structures, accordingly, the first issue to analyze is their load capacity to sustain axial loads, i.e., their stability and susceptibility to potential buckling collapse mechanisms that could compromise the structure and occupant safety. In the case of very slender columns, this refers to the problem of stability in elastic states.

The basic theory of slender rods losing stability in elastic states, as known, has been originally formulated by Euler (1744). He first introduced the concept of critical load P_{cr} and presented, according to his theory, the differential equation of an elastic deflected central line.

The stability phenomenon of semi slender columns in elastic-plastic states was researched too, by (Tetmajer, 1886; Jasiński, 1894; Engesser, 1889; 1895; Ostenfeld, 1898; Kármán, 1908; 1910; Shanley, 1947; Stowell, 1948; Bijlaard, 1949; Bleich, 1952; Broszko, 1953; Ylinen, 1956; Radhakrishnan, 1956; Gerard and Becker, 1957; Gerard, 1957; 1962; Seide *et al.*, 1960; Vol'mir, 1965; Březina, 1966).

Review of the Literature

This phenomenon was later researched by others.

Yiotis *et al.* (1982) presented a solution methodology for investigating the stability of rectangular box-shaped structures subjected to transverse uniformly distributed compressive loading. Nakashima *et al.* (1994) presented the results of a pilot test conducted for evaluating the energy dissipation behavior of shear panels made of low yield steel whose 0.2% offset yield stress is 120 MPa. Brank *et al.* (1997) presented a large-deformation model for thin shells composed of elastic-plastic material. Lepik (1999) considered a bifurcation of axially loaded elastic-plastic cylindrical shells in the case of an axisymmetric buckling. Papanastasiou and Durban (1999) presented a linear bifurcation analysis for pressure-sensitive elastic-plastic hollow cylinders under radial surface loads. Dubina and Ungureanu (2000) dealt with the elastic-plastic interactive buckling of thin-walled steel compression members. Abdel-Lateef *et al.* (2001) presented the elastic stability analysis of a column with variable cross-section subjected to distributed and concentrated axial load. Lilkova-Markova and Dzhupanov (2001) dealt with the dynamic stability of short continuous pipes conveying liquid and supported by elastic supports. Milašinović *et al.* (2003) dealt with the buckling problem of steel columns using by Rheological-Dynamical Analogy (RDA). Seyranian and Privalova (2003) dealt with the optimization and post-buckling

behavior of columns elastically supported at both ends. Alvarenga and Silveira (2006) presented a study about the necessary steps to qualify a second-order inelastic analysis as an advanced one. D'Aniello *et al.* (2006) executed two full-scale experimental tests on the lateral load-displacement response of a Reinforced Concrete (RC) structure seismically retrofitted by buckling restrained braces. Lolov and Lilkova-Markova (2006) dealt with the dynamic stability of a curved pipe bent in the arc of a circle on the hinge supports at the ends. Fraldi *et al.* (2008) aimed at deriving assessment and design formulae for determining the elastic-plastic response and the ultimate compressive strength of circular concrete columns confined by the Fiber Reinforced Polymers (FRP). Sanchez and Salas (2008) dealt with seismic ground motions that cause large deformations of buried pipelines. Voyiadjis and Woelke (2008) presented a finite element model for the elastic-plastic and damage analysis of thin and thick shells. Wahrhaftig *et al.* (2008) evaluated a buckling critical load of bars subjected to their self-weight. Wahrhaftig *et al.* (2016) executed a calculation of the natural frequency of vibration and the stability verification of a slender column including the reducing effects of stiffness both of the axial force and creep. Wahrhaftig *et al.* (2019) executed an analytical determination of the vibration frequencies and buckling loads of slender reinforced concrete towers. Wahrhaftig *et al.* (2020a) evaluated a limit state of stress and strain of free-fixed columns with variable geometry according to criteria from the Brazilian code for concrete structures. Wahrhaftig *et al.* (2020b) did an evaluation of mathematical solutions for the determination of buckling of columns under self-weight. Wahrhaftig (2020) did a time-dependent analysis of slender, tapered reinforced concrete columns. Wahrhaftig *et al.* (2021) made a stress assessment in reinforcement for columns with concrete creep and shrinkage through Brazilian technical normative. Ismail (2011) provided an analysis of the dynamical behavior and stability of pipes conveying fluid. Phungpaingam *et al.* (2011) presented an alternative model to analyze the post-buckling behavior of a hinged-hinged column made from nonlinear material (i.e., Ludwick material). Beylergil *et al.* (2012) studied the buckling and compressive failure of adhesively-bonded stepped-lap joints (with/without composite patches) composed of pultruded glass fiber-reinforced polymer. Abed *et al.* (2013) presented the Finite-Element (FE) study of the axial load capacity of pre-twisted steel bars of rectangular cross-sections. Kambe *et al.* (2013) proposed a sandwich panel with plywood and steel members for a new structural member. Ananthi and Anbarasu (2014) studied the possibility of using built-up cold-formed steel columns composed of two-lipped channels interconnected using a series of batten plates.

Ananthi *et al.* (2021) using the Finite Element Model (FEM) previously reported a parametric study, comprising 132 models, described for stainless steel battened built-up columns. Eissa *et al.* (2014) analyzed the work of a saturation-based active controller for vibration suppression of a four-degree-of-freedom rotor-AMB system. Andreev and Tsybin (2015) gave the solution to the problem of the stability of a compressed rod with a variable cross-section.

Li *et al.* (2015) presented a novel scrimber composite. The attempts were made through theoretical analysis to predict the buckling stress of the column specimens under both elastic and inelastic buckling. In Fig. 9e they presented the graph of strains at mid-length of a slender column and measured by strain gauges similarly like in Fig. 4a in the book (Murawski, 2011a) and in Fig. 4 in the paper (Murawski and Kłos, 2007) and in the doctor's thesis (Murawski, 1999) as well as in Rys.2 in the paper (Murawski, 1992).

Patel *et al.* (2015) dealt with High-strength thin-walled Concrete-Filled Steel Tubular (CFST) columns widely used in modern composite structures that might undergo local and global buckling. Özbaşaran *et al.* (2015) presented an alternative design procedure for lateral-torsional buckling of the cantilever I-beams which aimed to simplify the calculation of critical loads and design moments. Al-Kamal (2016) presented the possible collapse mechanisms initiated by a precast flexural member dropping on a lower member.

Jakab *et al.* (2016) focused on load-bearing glass columns and also on the design, the load-bearing capacity and the stability issues of fins. In Fig. 4 they presented the graph of strains set at mid-length of a slender glass column measured by strain gauges similarly like in Fig. 4a in the book (Murawski, 2011a) and in Fig. 4 in the paper of (Murawski and Kłos, 2007) and in the doctor's thesis Murawski (1999) as well as in Rys.2 in the paper (Murawski, 1992). They described this as: "... Fig. 4 indicates the loading force vs. strains on the glass surface. At the beginning both outer surface of the glass column is in compression after that, the compression starts to decrease at one outer glass surface and tensile stresses develop. The buckling process starts during this phenomenon", i.e., is according to the Technical Stability Theory (TSTh).

Kalamara *et al.* (2016) executed an experimental investigation for the structural performance assessment of square hollow glass columns. Kukhar *et al.* (2016) formed a gradient curve of a temperature distribution of lengthwise of the billet by differentiated heating before profiling by buckling. Łukowicz *et al.* (2016) dealt with cold-formed steel sections as extensively affected the modern steel construction industry. Megahed (2016) dealt with steel-concrete composite columns used in modern buildings. He investigated the behavior of pin-

ended axially and eccentrically loaded concrete encased steel composite columns. Tarsha and Takla (2016) evaluated the ultimate load of composite columns "steel-concrete" having square or circular steel tubes filled hollow section with concrete. Abbas and Awazli (2017) developed a numerical model in a three-dimensional nonlinear finite element and then validated it against experimental results reported in the literature. Abdel-Karim *et al.* (2017) proposed a model for the strength analysis of High-Strength Concrete (HSC) columns subjected to eccentric loading. Ammash (2017) dealt with shape optimization of innovated cold-formed steel columns under uniaxial compressive loading.

Atteya *et al.* (2017) dealt with an axial load capacity and the stiffness of a rectangular Hollow Structural Section (HSS) of the steel tube. In Fig. 19 they presented the graphs of strains set at mid-length measured by strain gauge similarly like in Fig. 4a in the book Murawski (2011a) and in Fig. 4 in the paper (Murawski and Kłos, 2007) and in the doctor's thesis Murawski (1999) as well as in Rys.2 in the paper Murawski (1992). Those graphs showed the way of losing stability in accordance with the Technical Stability Theory.

Baru (2017) dealt with buckling, as the most prominent failure mode of steel column stability as well as the structural stability of steel structures. Johnson *et al.* (2017) reported the results of a numerical and theoretical study of the buckling phenomenon in elastic columns containing a line of holes. Bedon and Amadio (2017) did a unified approach for the buckling verification of structural glass elements. Bedon and Amadio (2018) they made a buckling analysis and design proposal for 2-side supported double Insulated Glass Units (IGUs) in compression. Oliveira *et al.* (2017) studied the shear effect on the buckling of columns embedded in an elastic medium, evidencing the interaction of the column with the foundation. Silvestre *et al.* (2018) studied the influence of the nature of the deformation mode (global, local and distortional) on the load, carrying capacity of beams beyond the yield load. The five beams with different cross-sections, lengths, supports and loadings were analyzed. Słowiński and Piekarczyk (2017) dealt with a safe and economic design of steel cylindrical shells according to European Standard EN 1993-1-6 often requiring a nonlinear analysis. Abdulazeez *et al.* (2018) presented a numerical study on the behavior of Hollow-Core Fiber Reinforced Polymer-Concrete-Steel (HC-FCS) columns under combined axial compression and lateral loadings. Brasil and Wahrhaftig (2018) did an experimental evaluation of the effect of geometric nonlinearities on structural resonances. Can *et al.* (2018) designed a novel crash box as a telescopic structure by joining coaxial tubes by using gradual bonding surface areas. Isleem *et al.* (2018) dealt with experimental and analytical investigations of the stress-strain behavior of

rectangular concrete columns externally confined with Carbon Fiber-Reinforced Polymer (CFRP) composites under axial compression loading. Lilkova-Markova and Lolov (2018) investigated the problem of loss of stability of an axially compressed column. The column was fixed at one of its ends and on transversal linear spring support at the other. Massumi *et al.* (2018) matched the real behavior of the RC structures constructed based on the assumed specifications of the used materials. Razdolsky (2018) focused on elastic stability analysis of battened columns and laced columns with crosswise, fir-shaped and serpentine lattices. Saeed and Eissa (2018) analyzed bifurcations of periodic motion of a horizontally supported nonlinear Jeffcott-rotor system having a transversely cracked shaft. Saeed and Eissa (2019) did a bifurcation analysis of a transversely cracked nonlinear Jeffcott-rotor system at different resonance cases. Saeed (2019) did an analysis of vibration behavior and motion bifurcation of a nonlinear asymmetric rotating shaft. Next (Saeed, 2020) did an analysis of the steady-state forward and backward whirling motion of the asymmetric nonlinear rotor system and (Saeed *et al.*, 2020a) did a nonlinear dynamic analysis of the six-pole rotor-AMB system under two different control configurations. Later (Saeed *et al.*, 2020b) executed an analysis of radial versus Cartesian control strategies to stabilize the nonlinear whirling motion of the six-pole rotor-AMBs and (Saeed *et al.*, 2020c) an analysis of periodic, quasi-periodic and chaotic motions diagnose a crack on a horizontally supported nonlinear rotor system. Next, (Saeed *et al.*, 2021a) made a sensitivity analysis and vibration control of asymmetric nonlinear rotating shaft system utilizing 4-pole AMBs as an actuator and (Saeed *et al.*, 2021b) did an analysis of the rub-impact forces between a controlled nonlinear rotating shaft system and the electromagnet pole legs.

Slimani *et al.* (2018) dealt with the concept of the effective length factor of columns representing an important parameter with regard to the elastic buckling analysis. Thumrongvut and Tiwjantuk (2018) presented the experimental results on the strength and axial behavior of rectangular steel tube columns filled with Cellular Lightweight Concrete (CLC) under axial compression. Anuntasena *et al.* (2019) presented the 3D Finite Element (FE) analysis of the Concrete-Encased Steel (CES) columns subjected to concentric or eccentric loadings. Ivanov (2019) studied small vibrations of a cylindrical shaft caused by inertial excitation. The shaft was vertically situated. It was supported by a spherical and a cylindrical joint. Krishan *et al.* (2019) presented a theoretical study of the structural resistance of compressed short concrete elements in a glass-fiber reinforced shell. Kudryavtsev (2019) presented the study of the behavior of axially loaded columns that consisted of two flanges and a thin triangularly corrugated web,

connected by automatic welding. Nazarimofrad and Shokrgozar (2019) dealt with a Buckling-Restrained Braced frame (BRB) as the seismic force-resisting system used in buildings. Nonlinear time history and incremental dynamic analysis techniques were applied to investigate the behavior of the two frames with different stories under different ground motion records.

Qi *et al.* (2019) dealt with the innovative pultruded Fiber Reinforced Polymer (FRP). Axial compression tests with both ends pinned were employed to investigate the columns under concentric load. Strain responses on specimens with different slenderness ratios were consistent with the observed failure modes. The courses of the values of the longitudinal strains in Fig. 10 correspond to the graph in Fig. 4 in the paper (Murawski and Kłos, 2007) and in Fig. 4a (Murawski, 2011b) - what confirms qualitatively the correctness of the Technical Stability Theory.

Roy *et al.* (2019) dealt with a built-up box section popular for column members in Cold-Formed Steel (CFS). Simão *et al.* (2019) presented a study on the buckling behavior of slender steel columns under fire conditions, which depended on two main factors: The thermal elongation of the column and the degradation of the steel mechanical properties due to temperature rise. Virgens *et al.* (2019) presented the experimental study of eccentrically loaded reinforced concrete columns with an added 35 mm self-compacting concrete jacket attached to the column's most compressed faces using wedge bolts. Zhou *et al.* (2019) introduced the effective length factor and imperfection factor to the current stability factor formula to calculate the ultimate load of the lattice boom accurately. Zucco *et al.* (2020) tested a 750×640×240 mm variable-stiffness unitized integrated-stiffener out-of-autoclave thermoplastic composite wing-box for a combined shear-bending-torsion induced buckling load. Abedini *et al.* (2020) focused on investigating blast load parameters to design of Reinforced Concrete (RC) columns to withstand blast detonation. The numerical model was based on finite element analysis using LS-DYNA. Ahiwale *et al.* (2020) dealt with a Concrete-Filled Tubular (CFT) structure consisting of high strength, favorable ductility, fire resistance and energy absorption. Alomarah *et al.* (2020) dealt with auxetic structures that exhibit Negative Poisson's Ratio (NPR). Avci-Karatas (2020) dealt with construction in areas of high earthquake intensity, extreme climates and blast loading. Doan *et al.* (2020) dealt with a design of thin-walled composite columns. Goroshko *et al.* (2020) proposed a method of preventing the loss of Euler stability by thin rods. Kiss (2020a) investigated the planar stability of fixed-fixed shallow circular arches and (Kiss, 2020b) aimed to find the buckling loads for pinned-rotationally restrained shallow circular arches in terms of the rotational end stiffness,

geometry and material distribution. Naseri *et al.* (2020) presented an experimental study into the buckling behavior of Glass Fabric-Reinforced Polymer (GFRP) cylindrical shells subjected to axial compression load. Pinarbasi *et al.* (2020) dealt with the Turkish Building Code for Steel Structures replaced with a more rational Specification of Design and Construction of Steel Structures (SDCSS), which was prepared based on the American steel design specification (AISC 360-16). Qays and Al-Zuhairi (2020) discussed the idea of using slender Reinforced Concrete (RC) columns with cross-shaped (+-shaped) instead of square-shaped columns. Rajkannu and Sanjeevi (2020) presented the details of an experimental and numerical study on the effect of warping on the Flexural-Torsional Buckling (FTB) behavior of axially loaded cold-formed steel lipped channel members. Saberi *et al.* (2020) studied the cooperation of steel and concrete in composite columns. Saingam *et al.* (2020) dealt with a seismically retrofitting Reinforced Concrete (RC) building with a combination of Buckling-Restrained Braces (BRBs) and elastic steel frames that provided additional lateral stiffness and energy dissipation capacity. Terazawa *et al.* (2020) dealt with a grid-purlin system composed of RHS members known to be effective to prevent buckling of the welded beams. Viana *et al.* (2020) addressed a corotational Lagrangian formulation for nonlinear dynamic analysis of steel planar frames. Mehrabi *et al.* (2021) studied the dynamic response and mechanical performance of fiber-reinforced concrete columns using hybrid numerical algorithms.

Besides, there was done the literature review of semi-slender, thin-walled column stability (Murawski 2008a; 2008b; 2008c; 2008d; 2008e; 2020a; 2020b; 2020c).

Stability of Columns in Elastic-Plastic States

An application of thin-walled columns for structures mainly depends on their load capacity for axial loads, i.e., their stability. In the case of slender columns, this will refer to the stability in elastic states, but more often in engineering practice in the elastic-plastic states. In an analysis of stability in practical designing for squat columns, the determining of critical force may be used by a simplification formulated by (Tetmajer, 1886; Jasiński, 1894, 1985).

The simplification relies on the replacement of Euler's hyperbole by Tetmajer-Jasiński's straight line. For materials having the limit of plastic stress $\sigma_{pl} = R^*_e$ and limit of elastic stress $\sigma_H = R^{E_{th}}$ the co-ordinates of Tetmajer-Jasiński's straight line are: $\sigma_{cr} = \sigma_H = R^{E_{th}}$ for the slenderness ratio limiting elastic states $\lambda = \lambda_{el_lt}$ (simultaneously on the Euler's hyperbole) and $\sigma_{cr} = \sigma_{pl} = R^*_e$ for $\lambda = 0$ and in that case, the formula of Tetmajer-Jasiński's straight line is as follows:

$$\sigma_{cr}^{T-J} = R_e^* - \frac{R_e^* - R_H^{Eu}}{\lambda_{el_lt}} \cdot \lambda, \quad (1)$$

it may be also presented as depending on (L^*t) and A .

For squat thin-walled cylindrical columns (Fig. 1) it is as follows:

$$\sigma_{cr_cylindr}^{T-J} = R_e^* - 2\sqrt{2} \cdot \pi \cdot \frac{(R_e^* - R_H^{Eu}) \cdot (L \cdot t)}{\lambda_{el_lt} \cdot A}, \quad (2)$$

and for squat thin-walled square columns (Fig. 2):

$$\sigma_{cr_square}^{T-J} = R_e^* - 4\sqrt{6} \cdot \frac{(R_e^* - R_H^{Eu}) \cdot (L \cdot t)}{\lambda_{el_lt} \cdot A}. \quad (3)$$

Tetmajer-Jasiński's simplification was described in the paper of (Murawski, 2008a).

The next simplification which may be used in the analysis of stability for semi-slender columns to determine the critical force in practical designing is the one formulated by Ostenfeld (1898) and Johnson.

The simplification relies on the replacement of Euler's hyperbole by Johnson-Ostenfeld's parabola.

For materials having the limit of plastic stress $\sigma_{pl} = R_e^*$ the critical force $\sigma_{cr} = \sigma_{pl} = R_e^*$ for $\lambda = 0$ and the formula of Johnson-Ostenfeld's parabola is as follows:

$$\sigma_{cr}^{J-O} = R_e^* - \frac{R_e^{*2}}{4 \cdot E \cdot \pi^2} \cdot (\lambda)^2. \quad (4)$$

Johnson-Ostenfeld's parabola for squat thin-walled cylinder columns depending on (L^*t) and A is as follows (Fig. 3):

$$\sigma_{cr_cylindr}^{J-O} = R_e^* - \frac{2 \cdot R_e^{*2}}{E} \cdot \left[\frac{(L \cdot t)}{A} \right]^2, \quad (5)$$

and for squat thin-walled square columns (Fig. 4):

$$\sigma_{cr_square}^{J-O} = R_e^* - \frac{24 \cdot R_e^{*2}}{\pi^2 \cdot E} \cdot \left[\frac{(L \cdot t)}{A} \right]^2. \quad (6)$$

Johnson-Ostenfeld's simplification was described in the paper Murawski (2008b).

Ylinen (1956) used the approximation of the function $E_t = d\sigma/d\varepsilon = E_t(\sigma)$ and obtained the equation:

$$\sigma_{cr}^{Ylinen} = \frac{2 \cdot R_e}{1 + \left(\frac{\lambda}{\pi} \right)^2 \frac{R_e}{E} + \sqrt{\left[1 + \left(\frac{\lambda}{\pi} \right)^2 \frac{R_e}{E} \right]^2 - 4 \cdot c \cdot \left(\frac{\lambda}{\pi} \right)^2 \frac{R_e}{E}}} \quad (7)$$

For cylindrical columns, the critical stress according to Ylinen's formula depending on (L^*t) and A equals (Fig. 5):

$$\sigma_{cr_cylindr}^{Ylinen} = \frac{2 \cdot R_e^*}{1 + \left[\frac{2\sqrt{2} \cdot (L \cdot t)}{A} \right]^2 \frac{R_e^*}{E} + \sqrt{\left\{ 1 + \left[\frac{2\sqrt{2} \cdot (L \cdot t)}{A} \right]^2 \frac{R_e^*}{E} \right\}^2 - 4 \cdot c \cdot \left[\frac{2\sqrt{2} \cdot (L \cdot t)}{A} \right]^2 \frac{R_e^*}{E}}} \quad (8)$$

and for square columns (Fig. 6) is equal to:

$$\sigma_{cr_square}^{Ylinen} = \frac{2 \cdot R_e^*}{1 + \left[\frac{4\sqrt{6} \cdot (L \cdot t)}{\pi \cdot A} \right]^2 \frac{R_e^*}{E} + \sqrt{\left\{ 1 + \left[\frac{4\sqrt{6} \cdot (L \cdot t)}{\pi \cdot A} \right]^2 \frac{R_e^*}{E} \right\}^2 - 4 \cdot c \cdot \left[\frac{4\sqrt{6} \cdot (L \cdot t)}{\pi \cdot A} \right]^2 \frac{R_e^*}{E}}} \quad (9)$$

Ylinen's theory was described in the paper of (Murawski, 2008c).

Březina (1966) used the function $\sigma(\varepsilon)$ according to the DIN 4114 standard and obtained the equation:

$$\sigma_{cr}^{Brezina} = \frac{1}{2} \left\{ \left[2R_x - \frac{(R_e - R_x)^2}{E \cdot \left(\frac{\pi}{\lambda} \right)^2} \right] + \sqrt{\left[2R_x - \frac{(R_e - R_x)^2}{E \cdot \left(\frac{\pi}{\lambda} \right)^2} \right]^2 - 4 \cdot [R_x^2 - (R_e - R_x)^2]} \right\} \quad (10)$$

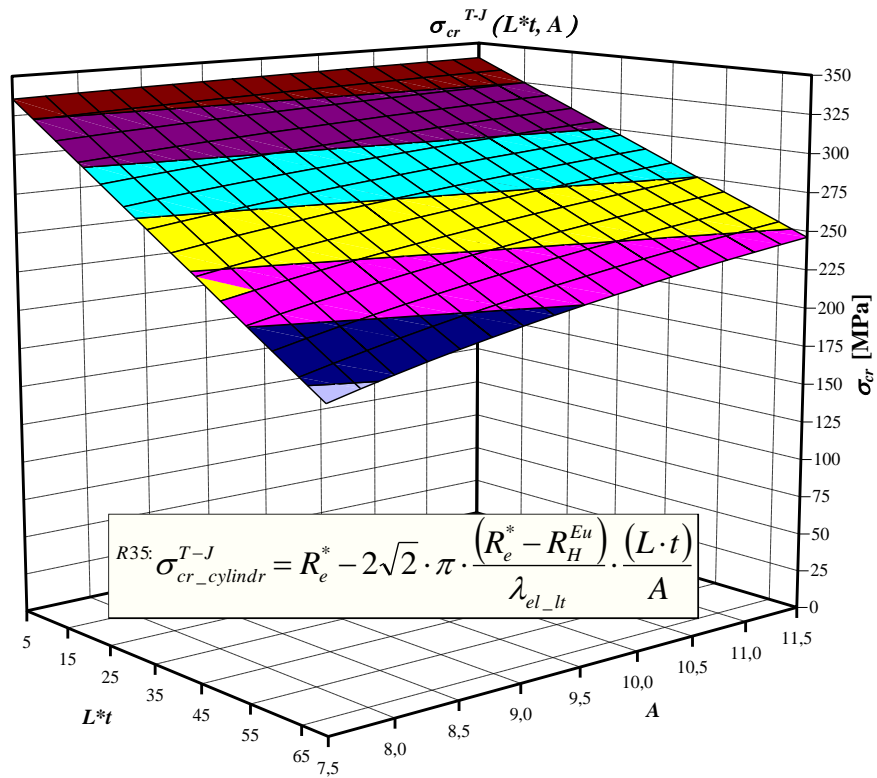


Fig. 1: Surface function $\sigma_{cr}^{T-J}_{cylindr}(L^*t, A)$ based on the Tetmajer-Jasiński formula of the cylindrical columns made of steel R35

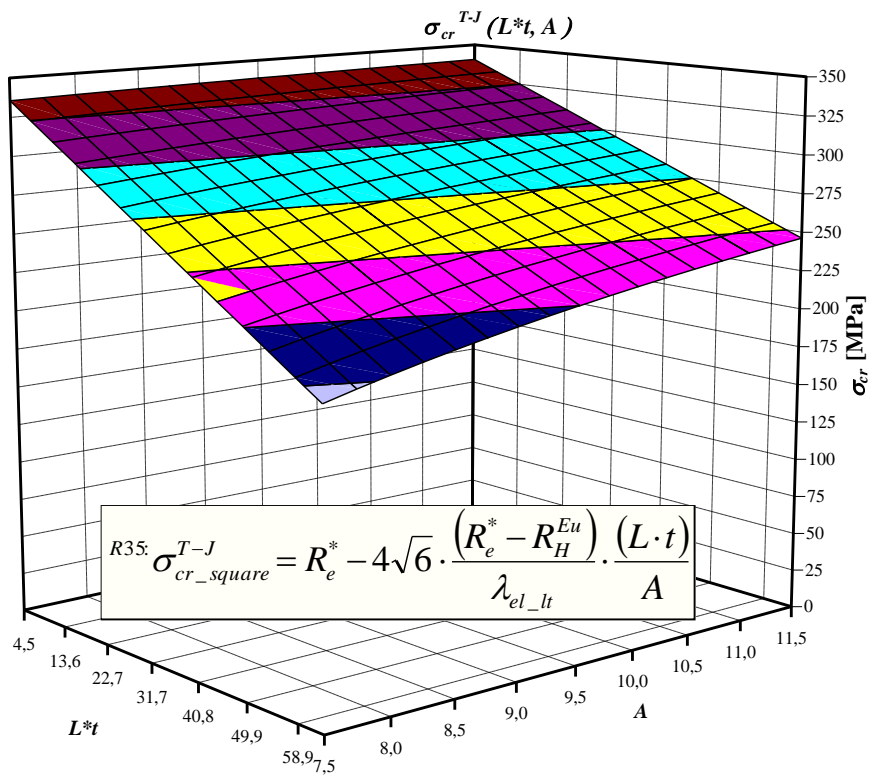


Fig. 2: Surface function $\sigma_{cr}^{T-J}_{square}(L^*t, A)$ based on the Tetmajer-Jasiński formula of the square-shaped columns made of steel R35

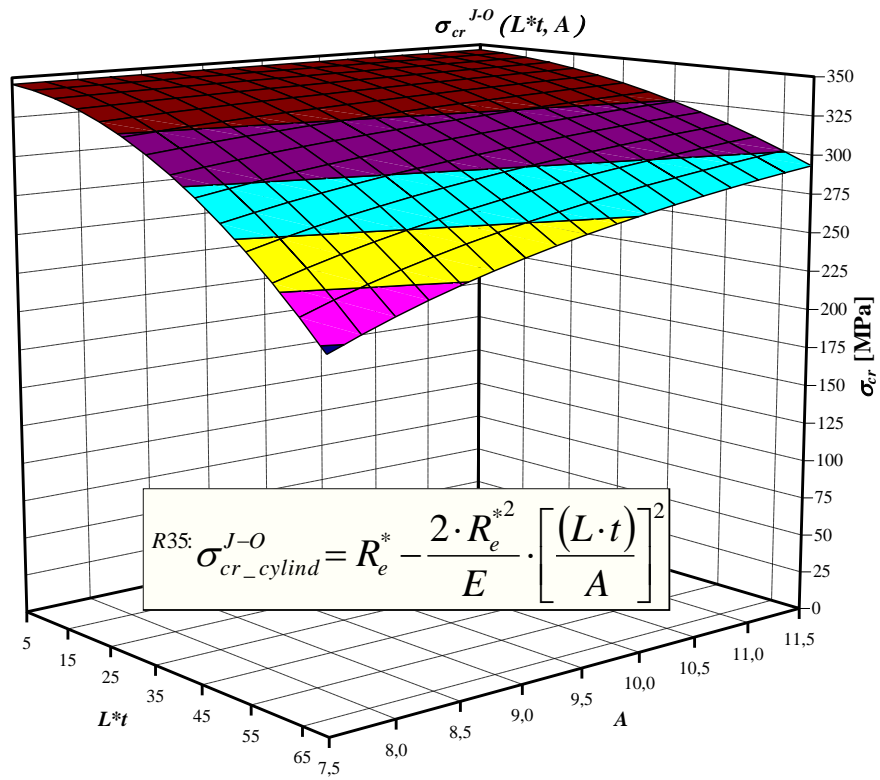


Fig. 3: Surface function $\sigma_{cr}^{J-O}_{cylind}(L*t, A)$ based on the Johnson-Ostenfeld formula of the cylindrical columns made of steel R35

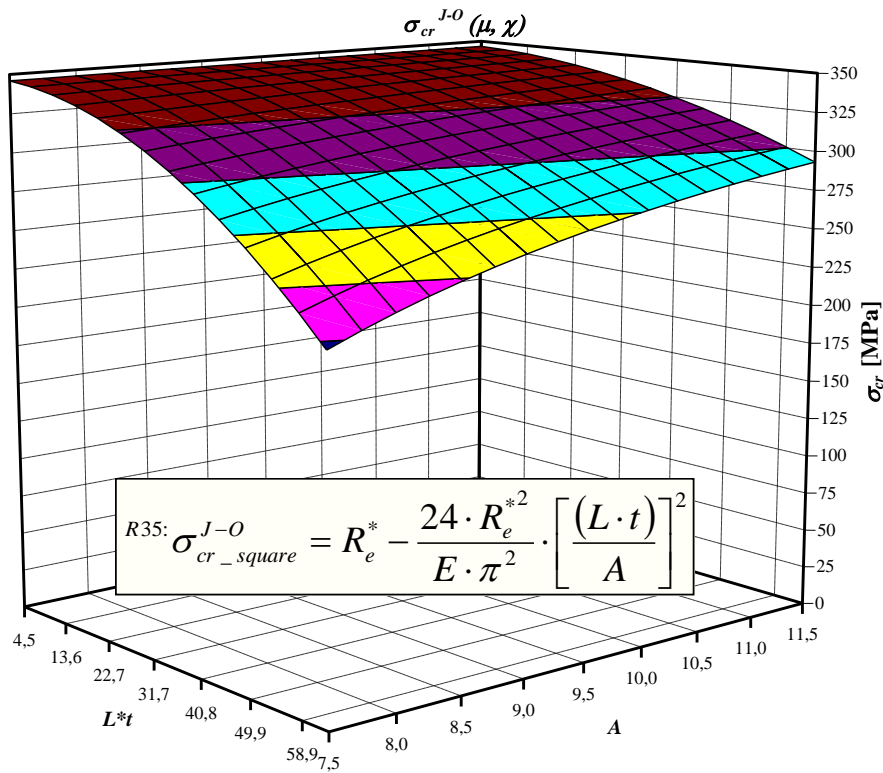


Fig. 4: Surface function $\sigma_{cr}^{J-O}_{square}(L*t, A)$ based on the Johnson-Ostenfeld formula of the square columns made of steel R35

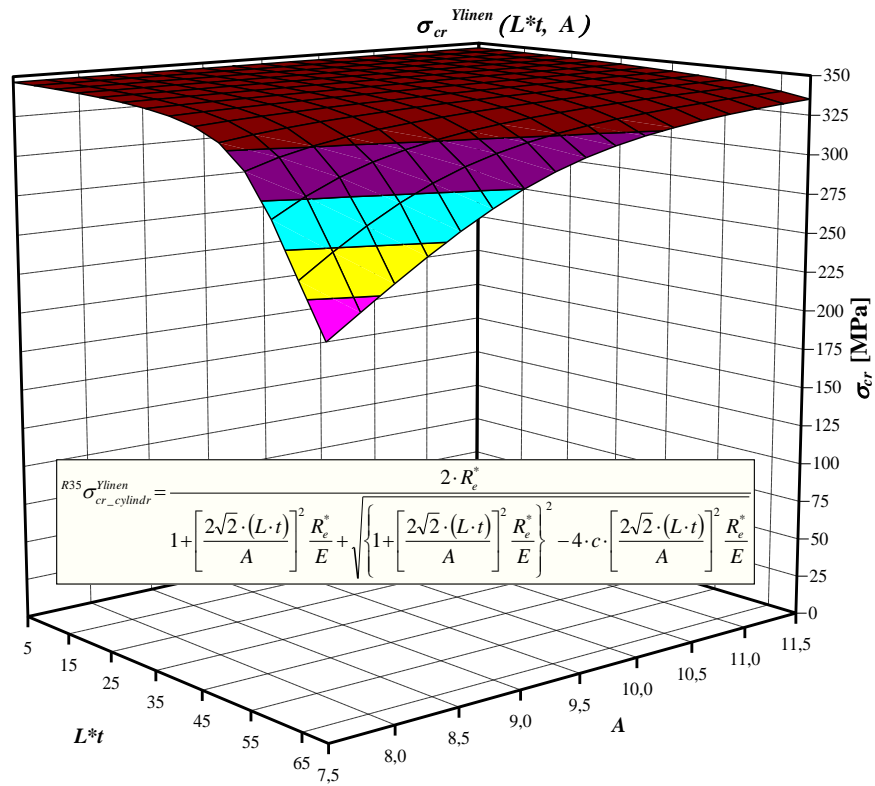


Fig. 5: Surface function $\sigma_{cr}^{Ylinen_{cylindr}}(L*t, A)$ based on the Ylinen's formula of the cylindrical shaped columns made of steel R35

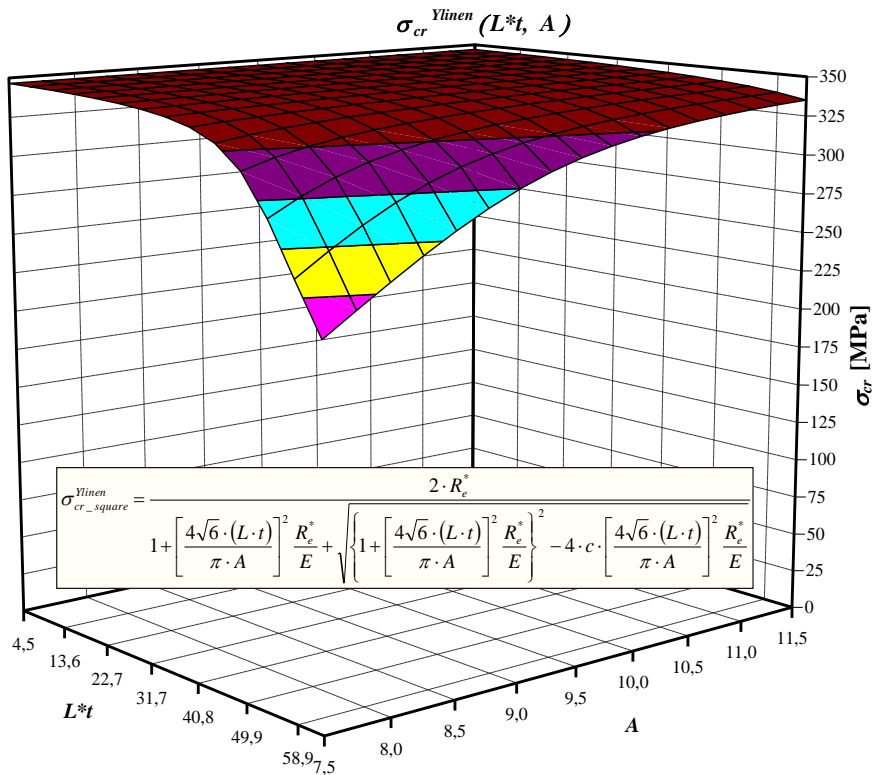


Fig. 6: Surface function $\sigma_{cr}^{Ylinen_{square}}(L*t, A)$ based on the Ylinen's formula of the square columns made of steel R35

For cylindrical columns the critical stress according to Březina (Fig. 7):

$$\sigma_{cr_cylinder}^{Březina} = \frac{1}{2} \left\{ \left[2R_H^{Eu} - \frac{(R_e^* - R_H^{Eu})^2}{E} \left[\frac{2\sqrt{2} \cdot (L \cdot t)}{A} \right]^2 \right]^2 + \sqrt{\left[2R_H^{Eu} - \frac{(R_e^* - R_H^{Eu})^2}{E} \left[\frac{2\sqrt{2} \cdot (L \cdot t)}{A} \right]^2 \right]^2 - 4 \cdot \left[R_H^{Eu2} - (R_e^* - R_H^{Eu})^2 \right]} \right\} \quad (11)$$

For the square columns, the critical stress according to Březina depending on $(L \cdot t)$ and A is equal to (Fig. 8):

$$\sigma_{cr_square}^{Březina} = \frac{1}{2} \left\{ \left[2R_H^{Eu} - \frac{(R_e^* - R_H^{Eu})^2}{E} \left[\frac{4\sqrt{6} \cdot (L \cdot t)}{\pi \cdot A} \right]^2 \right]^2 + \sqrt{\left[2R_H^{Eu} - \frac{(R_e^* - R_H^{Eu})^2}{E} \left[\frac{4\sqrt{6} \cdot (L \cdot t)}{\pi \cdot A} \right]^2 \right]^2 - 4 \cdot \left[R_H^{Eu2} - (R_e^* - R_H^{Eu})^2 \right]} \right\} \quad (12)$$

Březina's hypothesis was described in the paper of (Murawski, 2008d).

Pearson (1950; Bleich, 1952; Vol'mir, 1965) employed in their research the combination of the tangent modulus E_t and the modulus E . Using the function $\sigma(\varepsilon)$ according to the standard DIN 4114, they obtained the formula as follows:

$$\sigma_{cr}^{Pearson-Bleich-Vol'mir} = \frac{R_x + \sqrt{R_x^2 - \left[\frac{(R_e - R_x)^2 \lambda^4}{\pi^4 \cdot E^2} + 1 \right] \cdot \left[R_x^2 - (R_e - R_x)^2 \right]}}{\left[\frac{(R_e - R_x)^2 \lambda^4}{\pi^4 \cdot E^2} + 1 \right]} \quad (13)$$

For the cylindrical columns, the critical stress according to Pearson-Bleich-Vol'mir's formula depending on $(L \cdot t)$ and A is equal to (Fig. 9):

$$\sigma_{cr_cylinder}^{P-B-V} = \frac{R_H^{Eu} + \sqrt{R_H^{Eu2} - \left[\frac{(R_e^* - R_H^{Eu})^2 \cdot 64 \cdot (L \cdot t)^4}{A^4 \cdot E^2} + 1 \right] \cdot \left[R_H^{Eu2} - (R_e^* - R_H^{Eu})^2 \right]}}{\left[\frac{(R_e^* - R_H^{Eu})^2 \cdot 64 \cdot (L \cdot t)^4}{A^4 \cdot E^2} + 1 \right]} \quad (14)$$

and for the square columns (Fig. 10):

$$\sigma_{cr_square}^{P-B-V} = \frac{R_H^{Eu} + \sqrt{R_H^{Eu2} - \left[\frac{(R_e^* - R_H^{Eu})^2 \cdot 6^2 \cdot 4^4 \cdot (L \cdot t)^4}{\pi^4 \cdot A^4 \cdot E^2} + 1 \right] \cdot \left[R_H^{Eu2} - (R_e^* - R_H^{Eu})^2 \right]}}{\left[\frac{(R_e^* - R_H^{Eu})^2 \cdot 6^2 \cdot 4^4 \cdot (L \cdot t)^4}{\pi^4 \cdot A^4 \cdot E^2} + 1 \right]} \quad (15)$$

The loss stability theory of axially compressed slender columns in elastic-plastic states, based on the concept of the tangent modulus, was formulated by (Engesser, 1889; 1895; Kármán, 1908; 1910; Shanley, 1947).

Engesser-Kármán-Shanley's theory with the author's analysis of stability of thin-walled columns was described in the papers and books: Murawski (1999; 2002a; 2002b; 2003; 2008e; 2011a; 2011b; 2011c; 2017a; 2018).

In the case of stability of columns in elastic-plastic states, the author assumed that the loss of stability occurs already at minimum loads, whereas the position of the resultant neutral layer is changing what was caused by the superposition of pure compression and bending of the central line of the column.

This author's theory can be also named as the modified Engesser-Kármán-Shanley's theory and was described in the books: Murawski (2008e; 2011a; 2011b; 2011c; 2017a; 2018)

According to the assumption the state of stresses in the critical cross-section after the loss of stability and before the loss of carrying capacity results from the superposition of pure compression and bending. The formulas for the modified Engesser-Kármán-Shanley's critical stress are as follows:

$$\sigma_H^{KM}(\lambda) = R_H^{Eu} + \left(1 - \frac{\lambda}{\pi} \sqrt{\frac{R_H^{Eu}}{E}} \right) \left[R_H^* - R_H^{Eu} \right], \quad (16)$$

$$\sigma_{cr}^{KM}(\lambda) = \left(1 - \frac{\lambda}{\pi} \sqrt{\frac{R_H^{Eu}}{E}} \right) \left(R_e^* + R_H^* \frac{\lambda}{\pi} \sqrt{\frac{R_H^{Eu}}{E}} \right) + \frac{1}{E} \left(\frac{\lambda}{\pi} R_H^{Eu} \right)^2. \quad (17)$$

In the case of cylindrical columns according to the Engesser-Kármán-Shanley modified theory, the formula for the stress limiting elastic states is as (Fig. 12):

$$\sigma_{H_cylinder}^{KM} = R_H^{Eu} + \left[1 - \frac{2\sqrt{2} \cdot \pi \cdot (L \cdot t)}{\lambda_{el_it} \cdot A} \right] \cdot \left[R_H^* - R_H^{Eu} \right] \quad (18)$$

and the critical stress (Fig. 11):

$$\sigma_{cr_cylinder}^{KM} = R_H^{Eu} + \left[1 - \frac{2\sqrt{2} \cdot \pi \cdot (L \cdot t)}{\lambda_{el_it} \cdot A} \right] \cdot \left[R_H^* - 2R_H^{Eu} + R_e^* \right] + \left[1 - \frac{2\sqrt{2} \cdot \pi \cdot (L \cdot t)}{\lambda_{el_it} \cdot A} \right]^2 \cdot \left[R_H^* - R_H^{Eu} \right] \quad (19)$$

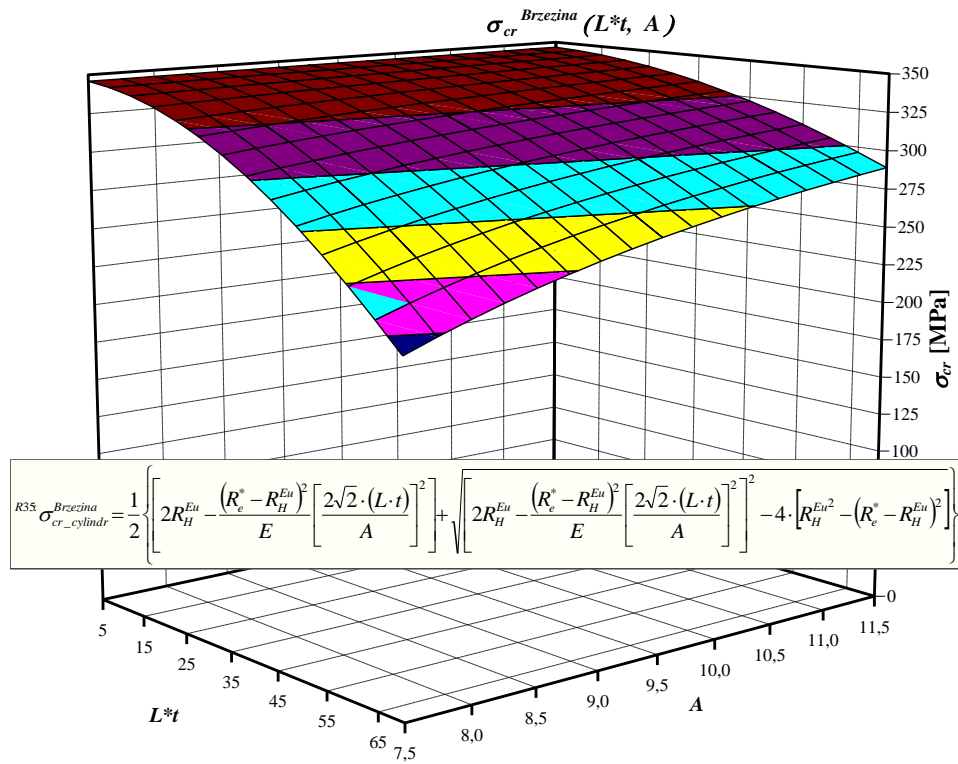


Fig. 7: Surface function $\sigma_{cr}^{Brezina}_{cylindr}(L*t, A)$ based on the Brezina's formula of the cylindrically-shaped columns made of steel R35

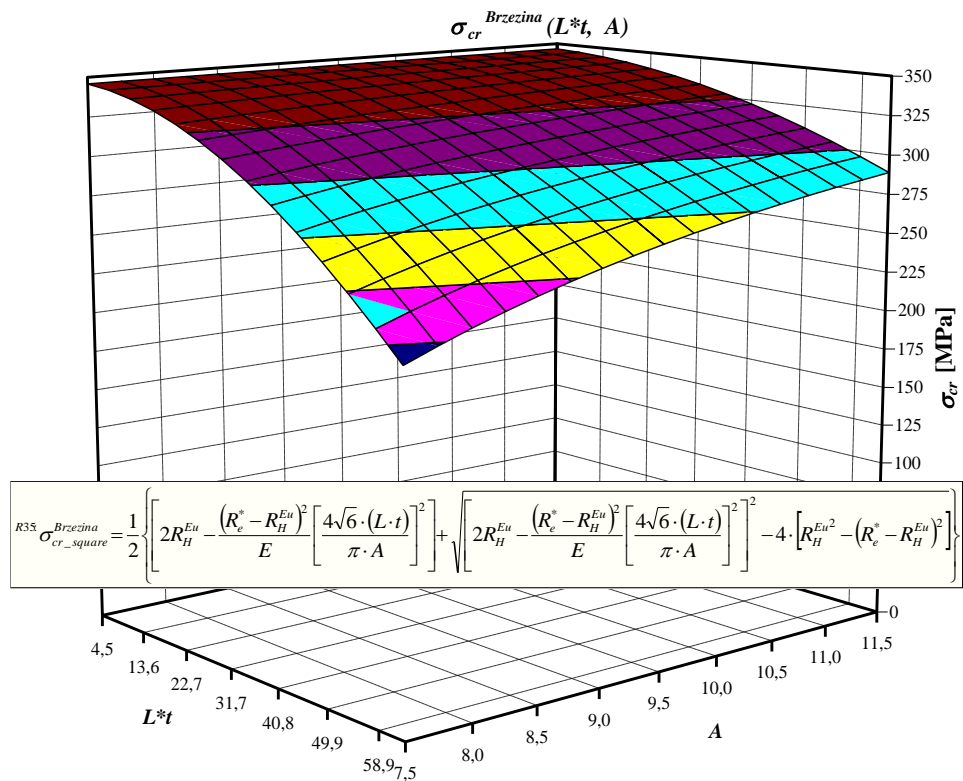


Fig. 8: Surface function $\sigma_{cr}^{Brezina}_{square}(L*t, A)$ based on the Brezina's formula of the square-shaped columns made of steel R35

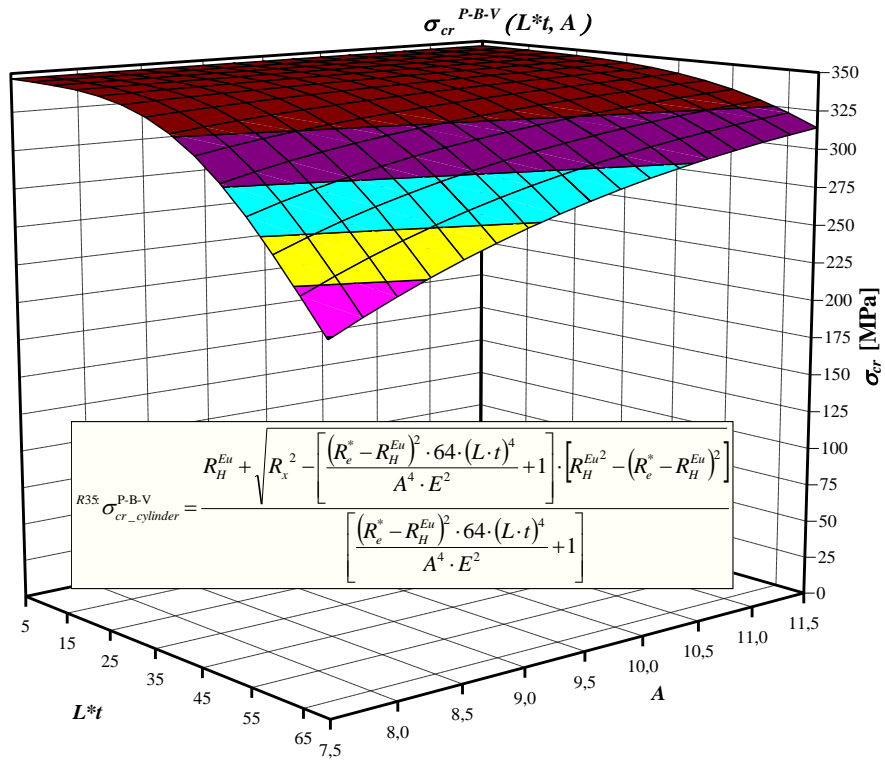


Fig. 9: Surface function $\sigma_{cr}^{P-B-V}_{cylinder}(L^*t, A)$ based on the Pearson-Bleich-Vol'mir's formula of the cylindrical columns made of steel R35

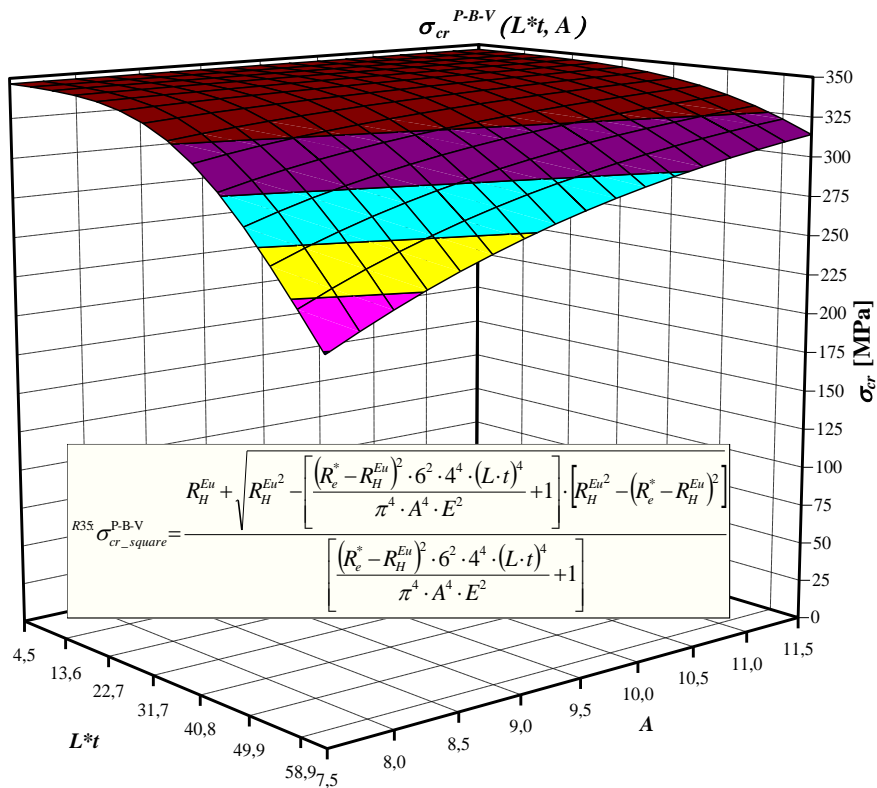


Fig. 10: Surface function $\sigma_{cr}^{P-B-V}_{square}(L^*t, A)$ based on the Pearson-Bleich-Vol'mir's formula of the square columns made of steel R35

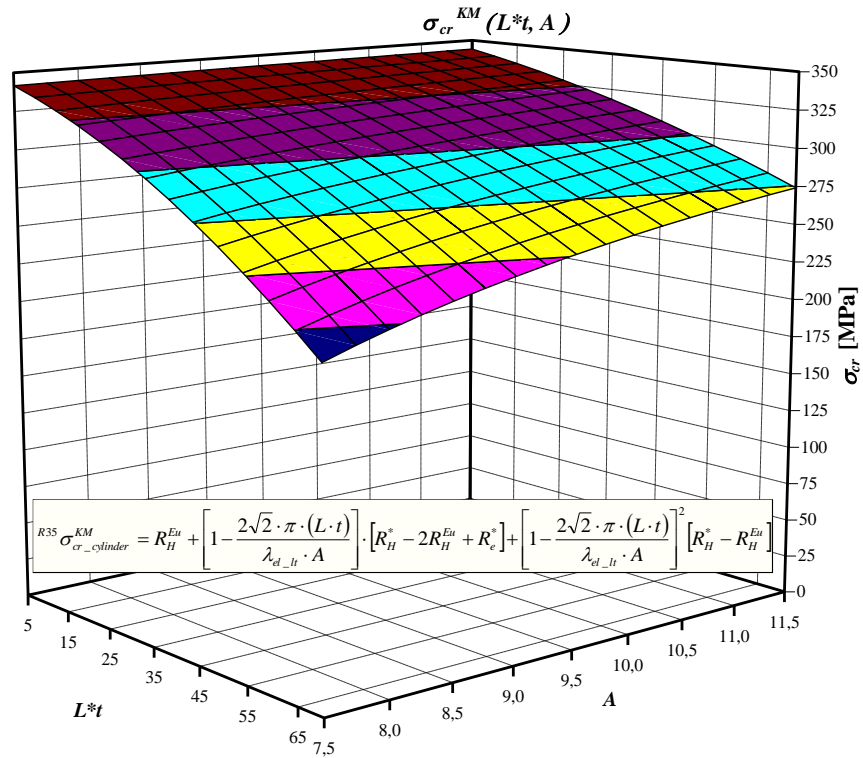


Fig. 11: Surface function $\sigma_{cr}^{KM}_{cylin}(L^*t, A)$ based on the modified Engesser-Kármán-Shanley’s formula of the cylindrical columns made of steel R35

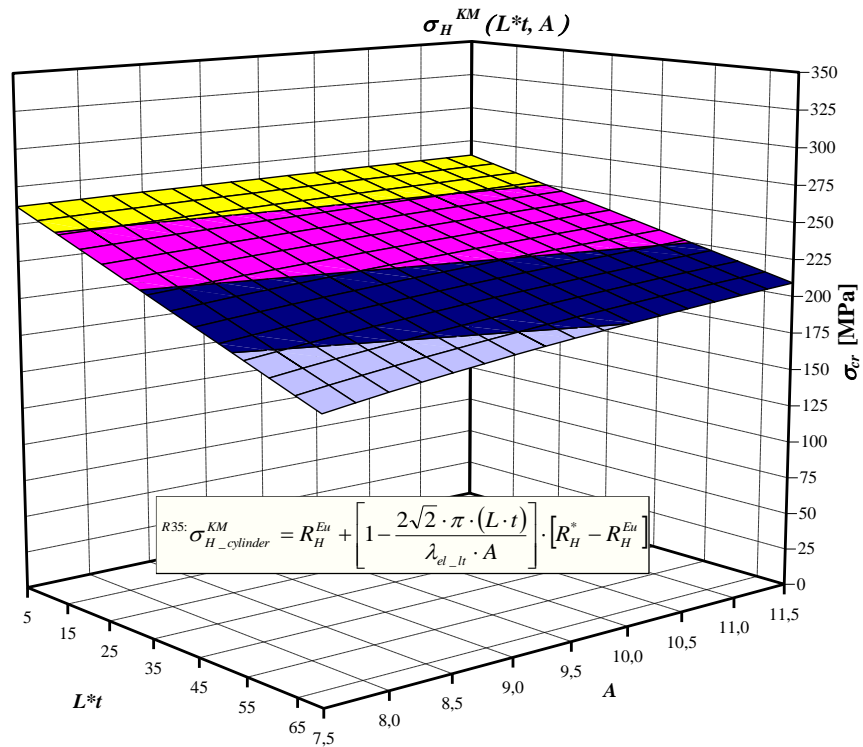


Fig. 12: Surface function $\sigma_H^{KM}_{cylin}(L^*t, A)$ based on the modified Engesser-Kármán-Shanley’s formula of the cylindrical columns made of steel R35

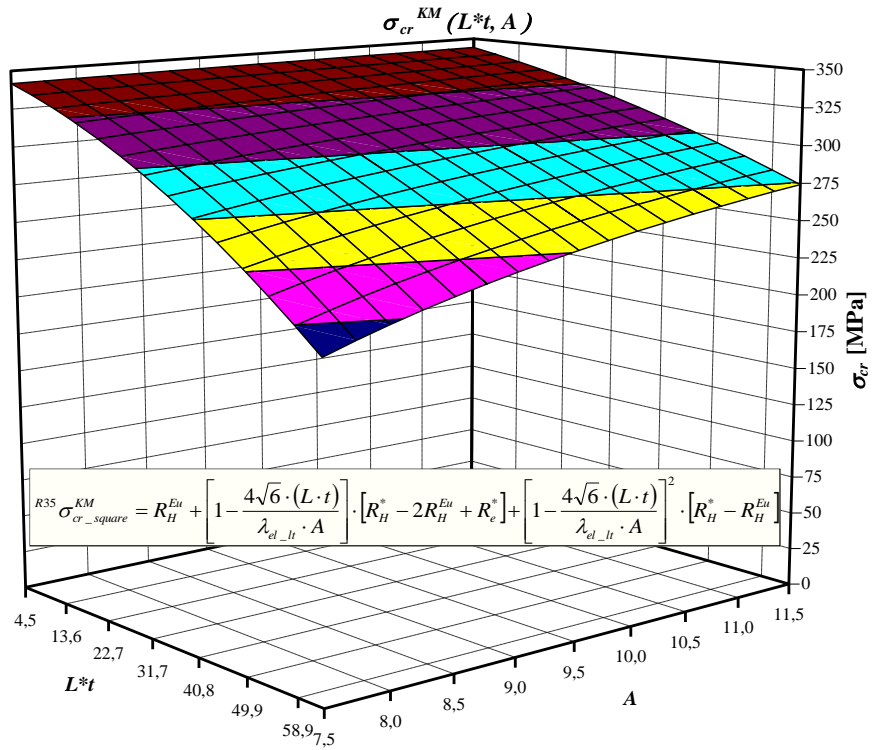


Fig. 13: Surface function $\sigma_{cr}^{KM_square}(L^*t, A)$ based on the modified Engesser-Kármán -Shanley's formula of the square columns made of steel R35

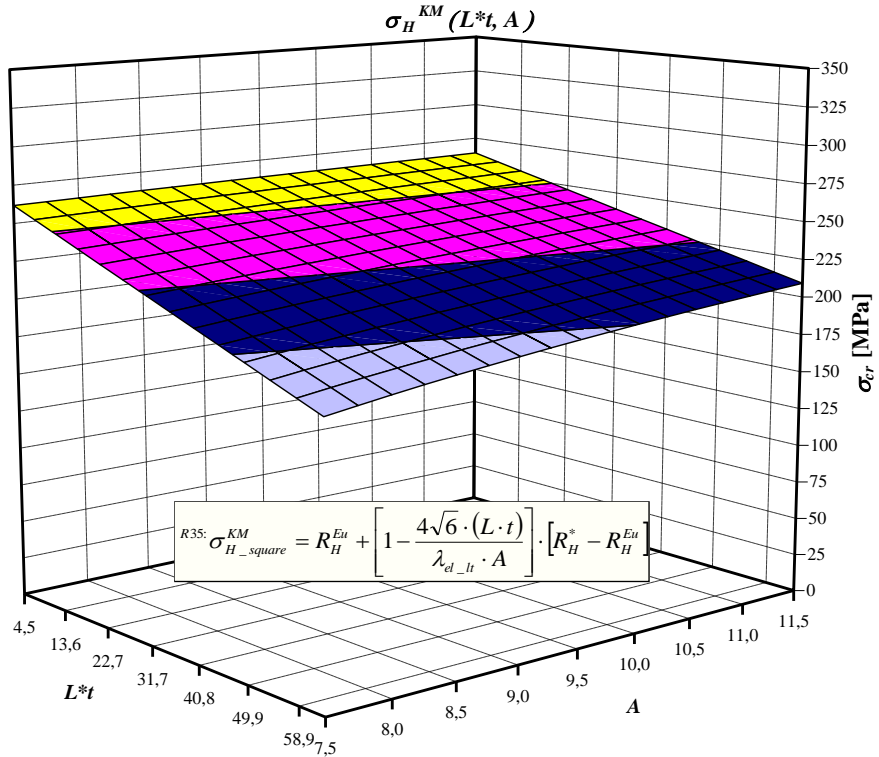


Fig. 14: Surface function $\sigma_H^{KM_square}(L^*t, A)$ based on the modified Engesser-Kármán-Shanley's formula of the square columns made of steel R35

In the case of an axially compressed square column by ball-and-socket joints, the elastic stress is as follows (Fig. 14):

$$\sigma_{H_square}^{KM} = R_H^{Eu} + \left[1 - \frac{4\sqrt{6} \cdot (L \cdot t)}{\lambda_{el-lt} \cdot A} \right] \cdot [R_H^* - R_H^{Eu}], \quad (20)$$

and the critical stress (Fig. 13):

$$\begin{aligned} \sigma_{cr_square}^{KM} = & R_H^{Eu} + \left[1 - \frac{4\sqrt{6} \cdot (L \cdot t)}{\lambda_{el-lt} \cdot A} \right] \cdot [R_H^* - 2R_H^{Eu} + R_c^*] \\ & + \left[1 - \frac{4\sqrt{6} \cdot (L \cdot t)}{\lambda_{el-lt} \cdot A} \right]^2 \cdot [R_H^* - R_H^{Eu}]. \end{aligned} \quad (21)$$

Experimental Research Works on Stability in Elastic-Plastic States of Columns Compressed Through Ball-and-Socket Joints with Friction

The tests of compression of specimens by applying an axial load through the steel ball-and-socket joints with friction (Fig. 15) were carried out using the test machine ZD 40 with the range of 40 kN. As the results of tests of the compressions of the specimens made of the same material were obtained the curves $P(\Delta a)$.

The curve obtained for a semi-slender column in the elastic-plastic states together with the type of the deformations is schematically presented in Fig. 16.

In order to show the differences in Fig. 17 is schematically presented the curve for a very slender column in elastic states, for which there are no latest deformations after relief a load.

Photo. 1 shows the specimen made of steel R35 with the cross-section $\phi 28 \times 1$ and the slenderness ratio $\lambda = 15$, compressed through ball-and-socket joints with so big partition of friction in the bottom joint, that part of the fold had appeared not in the middle of the column but at the upper end.

On the base of the experimental results executed on specimens made of steel R35 compressed through ball-and-socket joints with friction (Murawski, 1999; 2003; 2011c; 2017a) were determined the compress modulus E_c and secant compress modulus E_{sc} for thin-walled columns in elastic-plastic states (Fig. 16). They were determined analogically to Young's modulus E , tangent modulus E_t and secant modulus E_s during tension (Fig. 18). On the basis of the experimental results were determined the approximated functions: $\Delta L(\lambda)$ and $\varepsilon(\lambda)$ (Fig. 21 and 23), $P(\lambda)$ and $\sigma(\lambda)$ (Fig. 22, 24, 26 and 27) as well as $E_c(\lambda)$ and $E_{sc}(\lambda)$ (Fig. 25).

The approximated experimental functions $\sigma^{exp}(\lambda)$ were compared to the theoretical functions $\sigma^{KM}(\lambda)$ - Fig. 27.

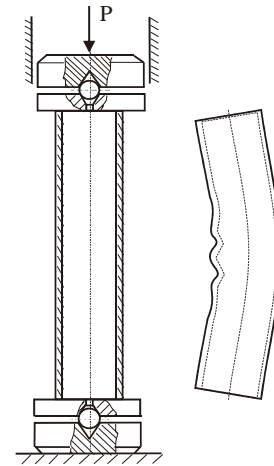


Fig. 15: Guidance and the fixing of the specimen during the compression through ball-and-socket joints with friction and the characteristic form of lateral buckling of the semi-slender cylindrical column in elastic-plastic states (Murawski, 2011a; 2011b; 2017a; 2020a)

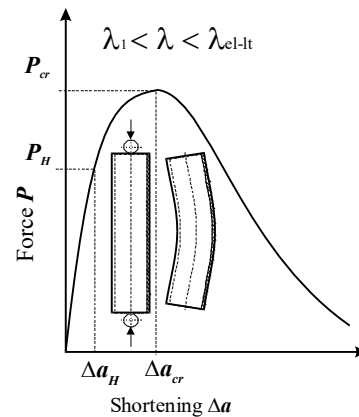


Fig. 16: Curve $P(\Delta a)$ of $\lambda_1 < \lambda < \lambda_{el-lt}$ in the elastic-plastic states (Murawski, 2011a; 2011b; 2017a)

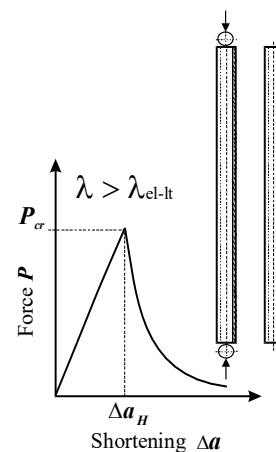


Fig. 17: Curve $P(\Delta a)$ of $\lambda > \lambda_{el-lt}$ in the elastic state (Murawski, 2011a; 2011b; 2017a)

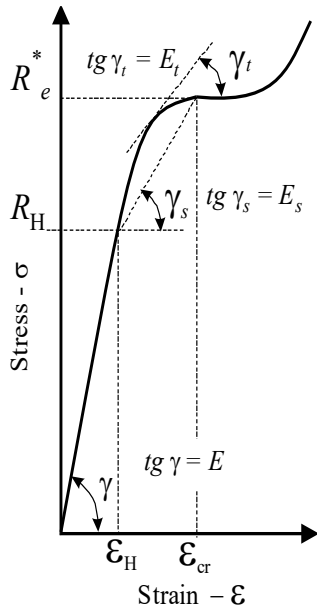


Fig. 18: Young's modulus E , tangent modulus E_t and secant modulus E_s during tension (Murawski, 2002a; 2002b; 2011a; 2011b; 2017a)



Photo 1: Specimen with the cross-section $\phi 28 \times 1$ and the slenderness ratio $\lambda = 15$, made of steel R35, compressed through ball-and-socket joints with the big partition of friction in the bottom joint (Murawski, 1999; 2011b; 2017a)

Discussion

The Tetmajer-Jasiński's surface functions (2) and (3) showed in Figs. 1 and 2 are almost linearly increasing with the transverse cross-section area A and strongly linearly decreasing with the L^*t product.

The Johnson-Ostenfeld's surface functions (5) and (6) showed in Figs. 3 and 4 are non-linearly increasing with the transverse cross-section area A and strongly non-linearly with second degree decreasing with the L^*t product.

The Ylinen's surface functions (8) and (9) showed in Figs. 5 and 6 are strong non-linear of increasing with the transverse cross-section area A and very strong parabolic non-linearly decreasing with the L^*t product, so big part of the surface functions are almost flat.

The Březina's surface functions (11) and (12) showed in Figs. 7 and 8 are non-linearly increasing with the transverse cross-section area A and strongly non-linearly with second degree decreasing with the L^*t product.

The Pearson-Bleich-Vol'mir's surface functions (14) and (15) showed in Fig. 9 and 10 are non-linearly increasing with the transverse cross-section area A and strongly parabolic non-linearly with second degree decreasing with the L^*t product.

The author's surface functions (18) and (20) for the limiting elastic stress showed in Figs. 12 and 14 are linearly decreasing with the L^*t product and linearly increasing the transverse cross-section area A , so the surface functions are flat.

The author's surface functions (19) and (21) for the critical compressive stress showed in Fig. 11 and 13 are slightly non-linearly decreasing with the L^*t product and slightly non-linearly increasing with the transverse cross-section area A , so a part of the surface function is almost flat.

In order to compare the experimental results to the results obtained from simplifications and hypotheses – the results in the case of columns with the transverse cross-section $\phi 50 \times 1$ and $\phi 28 \times 1$ made of the steel R35 were determined and showed for adequately ranges for elastic-plastic states as the graphs of the functions $\sigma_{cr}(\lambda)$ in Figs. 27-33.

The maximal departures from the experimental results of those obtained from simplifications and hypotheses are presented in Table 1 and 2.

The biggest maximal differences are between the experimental results and Ylinen's: -69,90 MPa and -21,63% for $\phi 50 \times 1$ and -68,93 MPa and -25,94% for $\phi 28 \times 1$.

The least maximal differences are between the experimental results and the author's approximated hypothesis: 7,06 MPa and 2,41% for $\phi 50 \times 1$ and -5,21 MPa and -2,01% for $\phi 28 \times 1$.

For the columns $\phi 50 \times 1$ on the base of the tests were assumed: $E^* = 195\,533$ MPa, $\lambda_{el-tl} = 102.69$, $R_e^* = 358.56$ MPa, $R_H^* = 247,34$ MPa, $R_H^{Eu} = 202.875$ MPa.

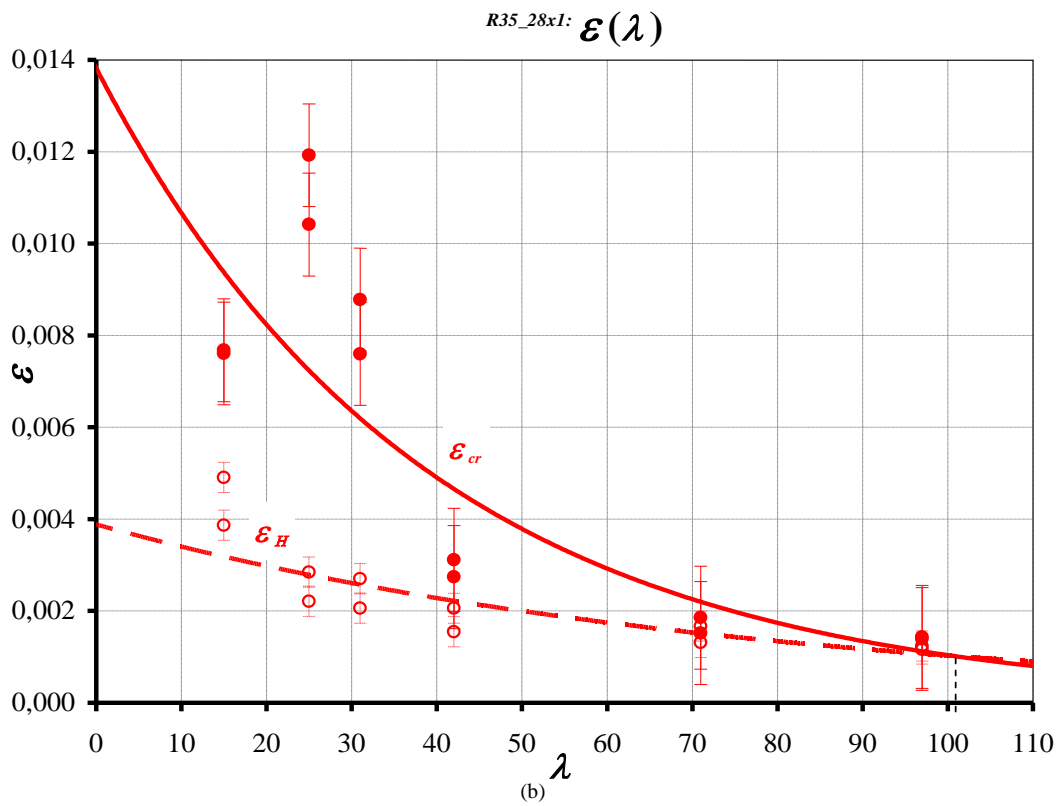
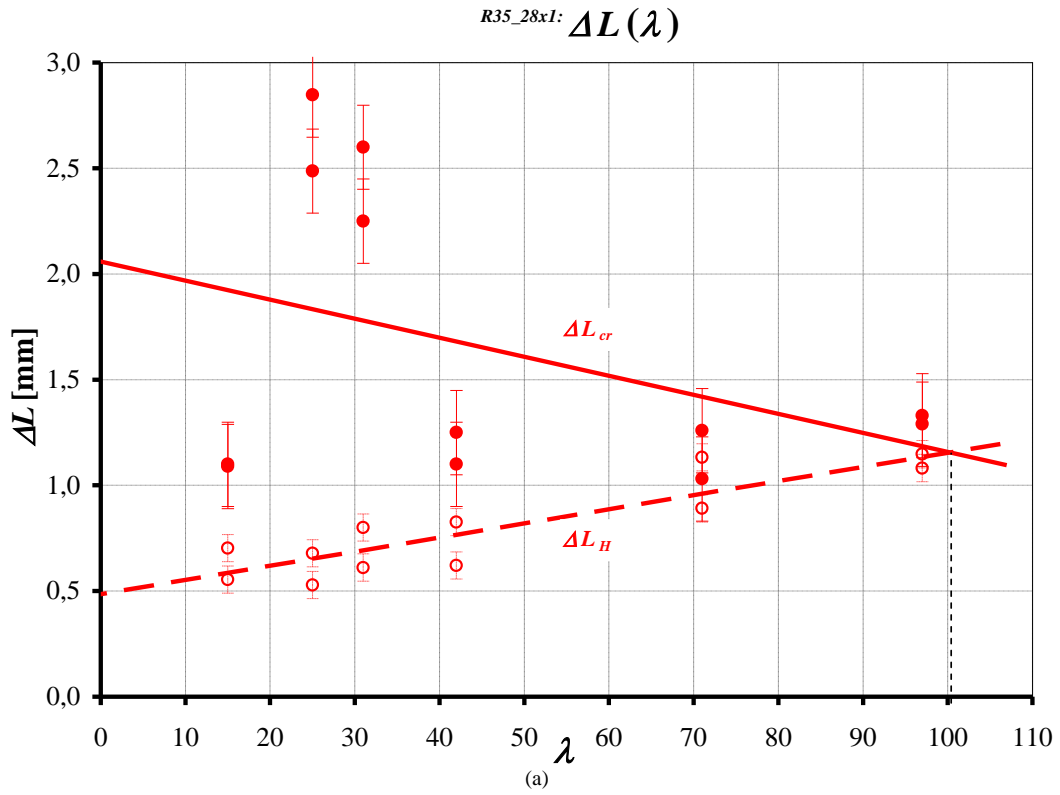


Fig. 21: Approximated functions: (a) $\Delta L(\lambda)$ and (b) $\varepsilon(\lambda)$ of the specimens with cross-section $\phi 28 \times 1$ made of steel R35, compressed through ball-and-socket joints with friction

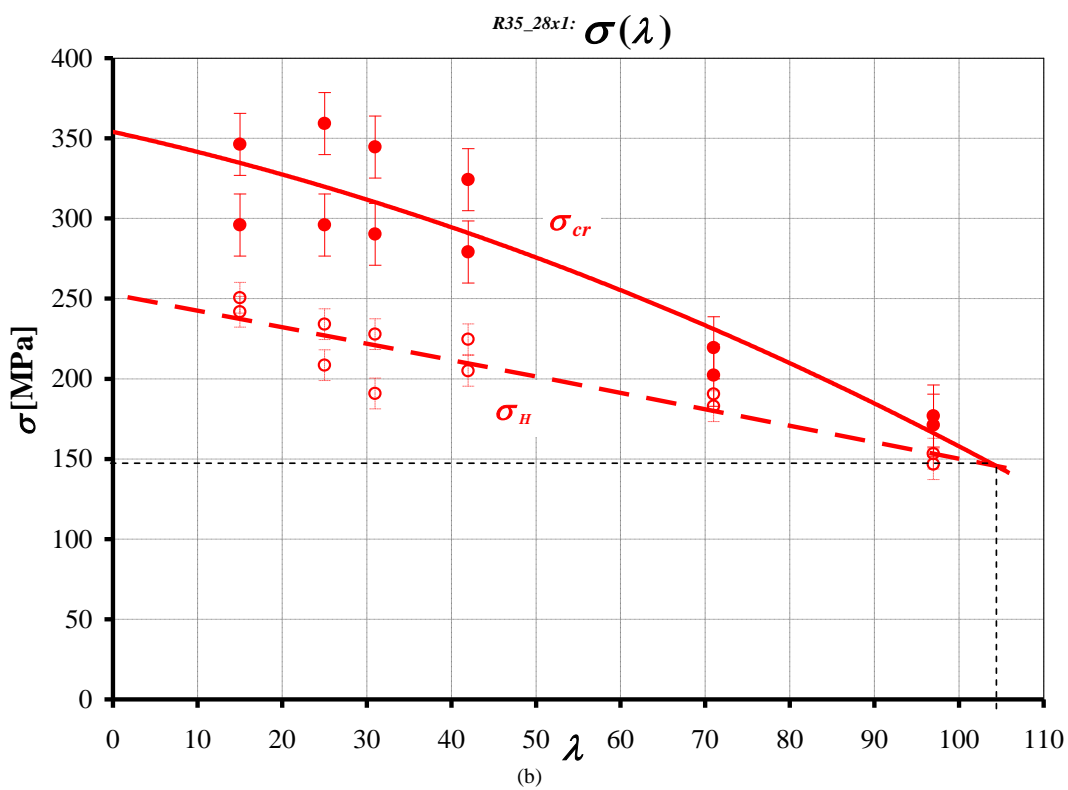
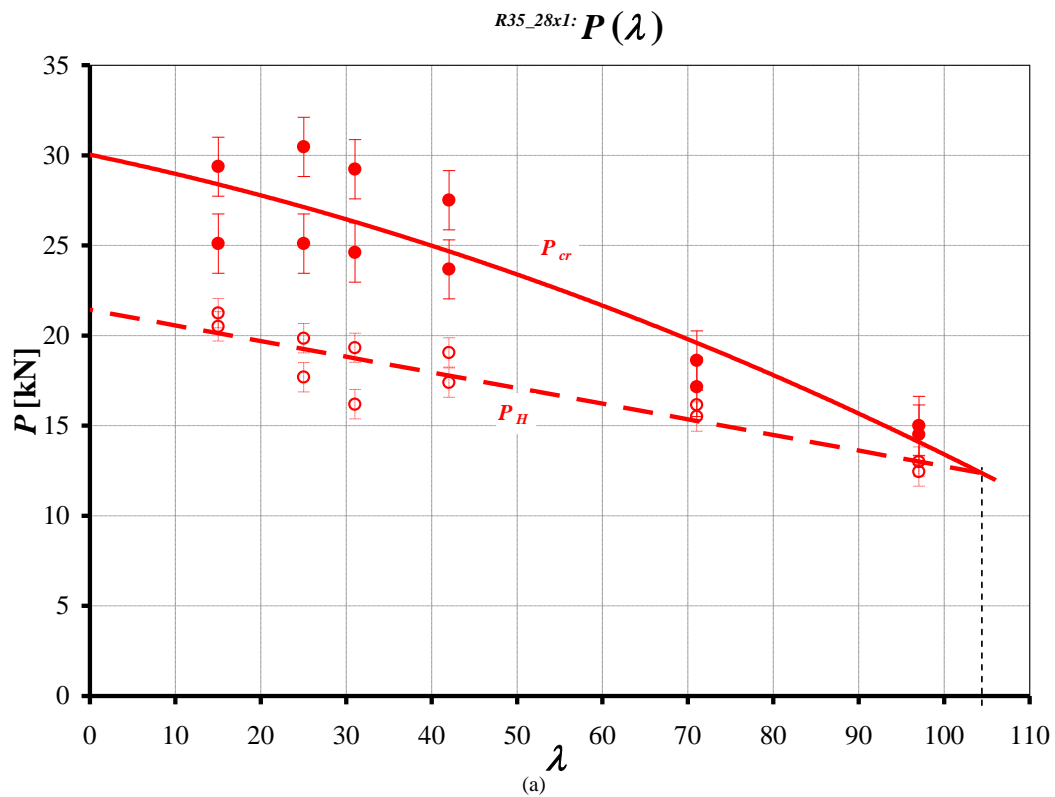


Fig. 22: Approximated functions: (a) $P(\lambda)$ and (b) $\sigma(\lambda)$ of the specimens with cross-section $\phi 28 \times 1$ made of steel R35, compressed through ball-and-socket joints with friction

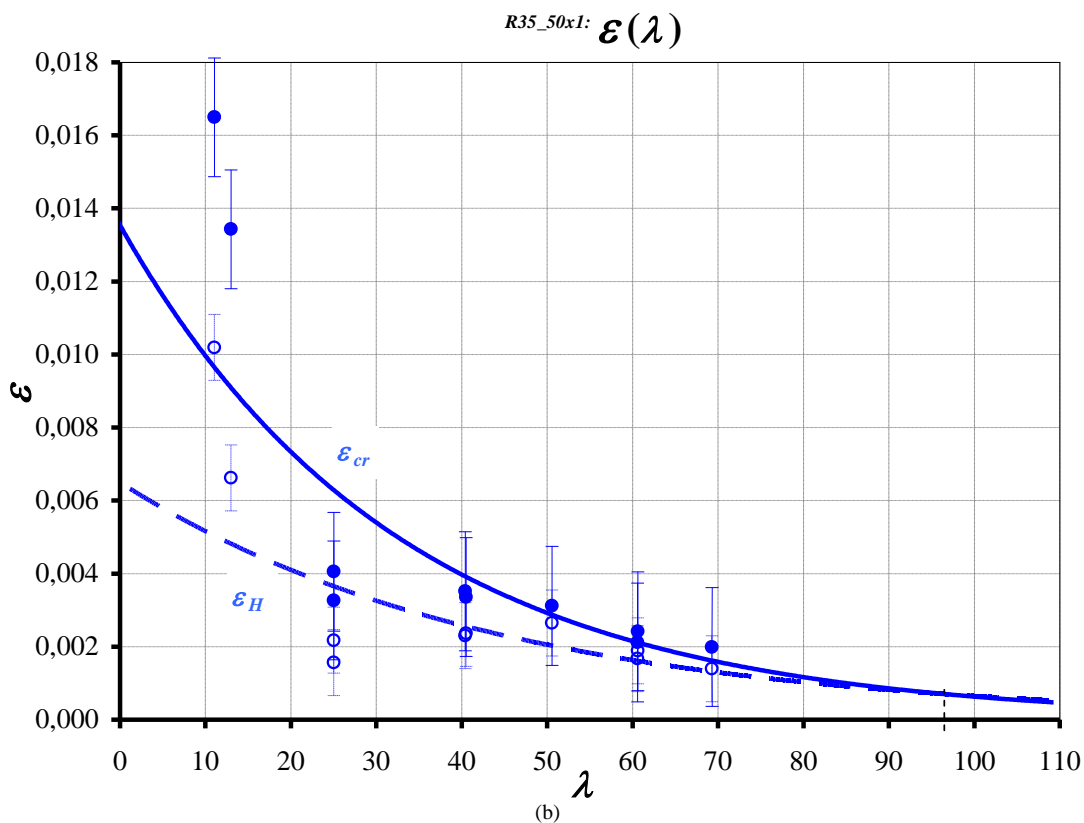
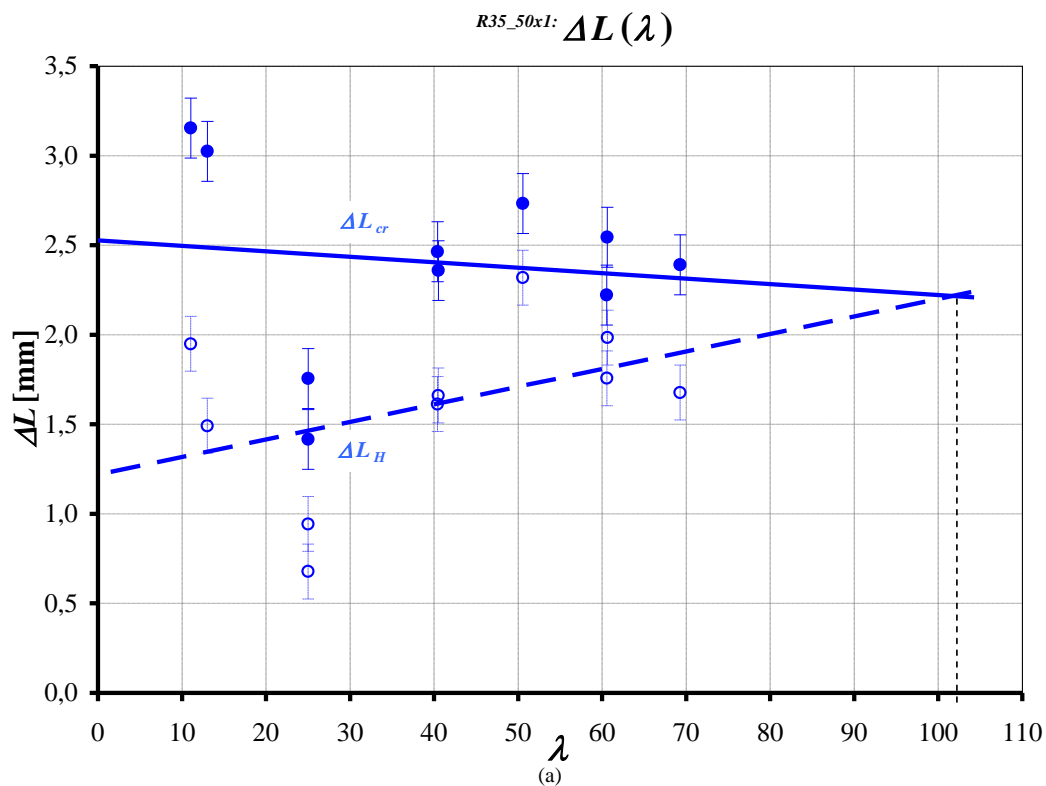


Fig. 23: Approximated functions: (a) $\Delta L(\lambda)$ and (b) $\varepsilon(\lambda)$ of the specimens with cross-section $\phi 50 \times 1$ made of steel R35, compressed through ball-and-socket joints with friction

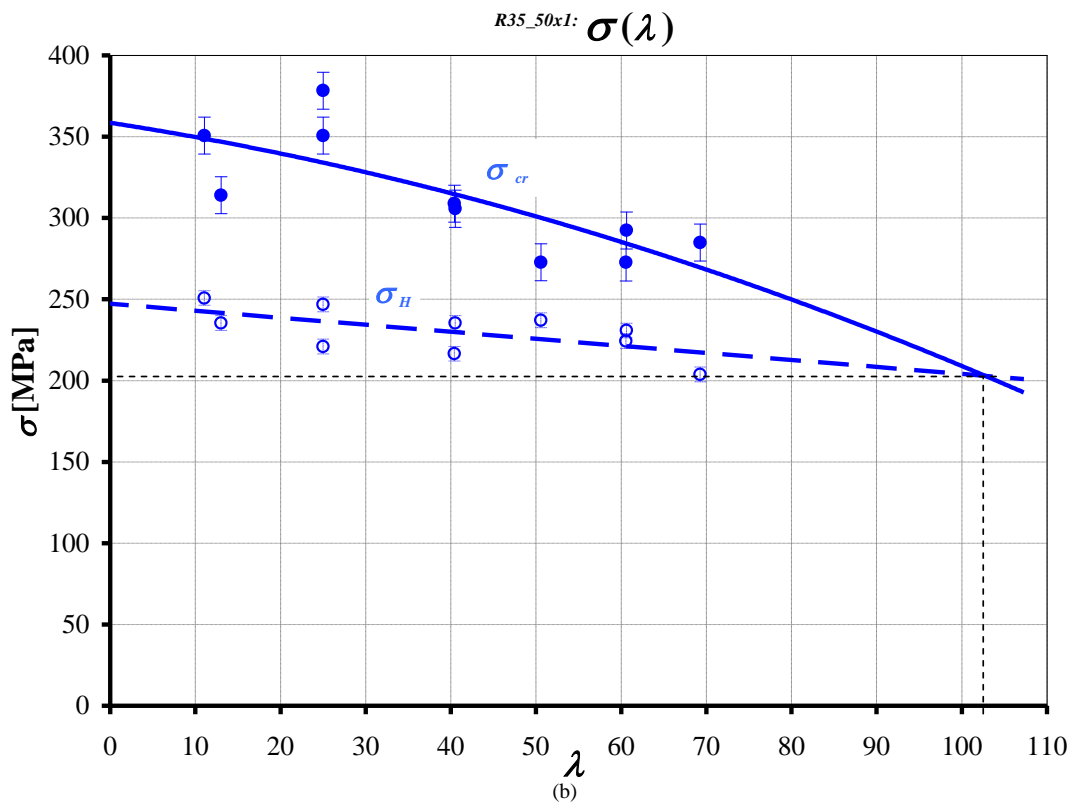
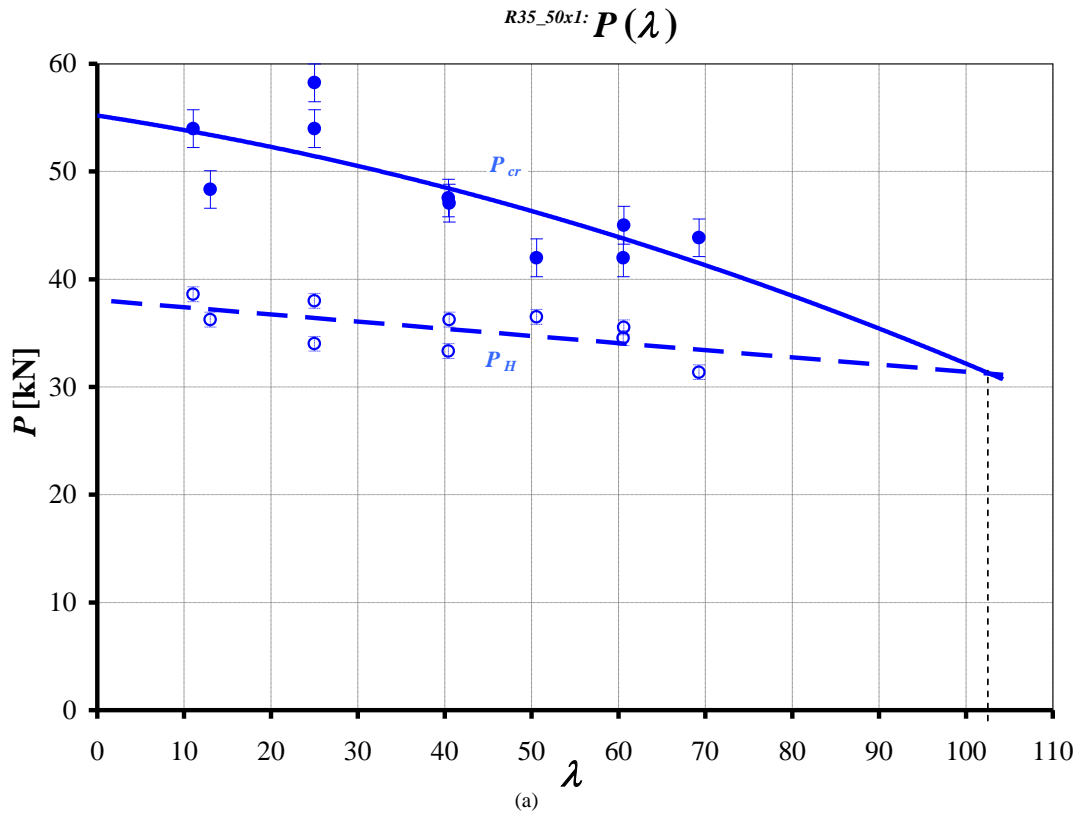


Fig. 24: Approximated functions: (a) $P(\lambda)$ and (b) $\sigma(\lambda)$ of the specimens with cross-section $\phi 50 \times 1$ made of steel R35, compressed through ball-and-socket joints with friction

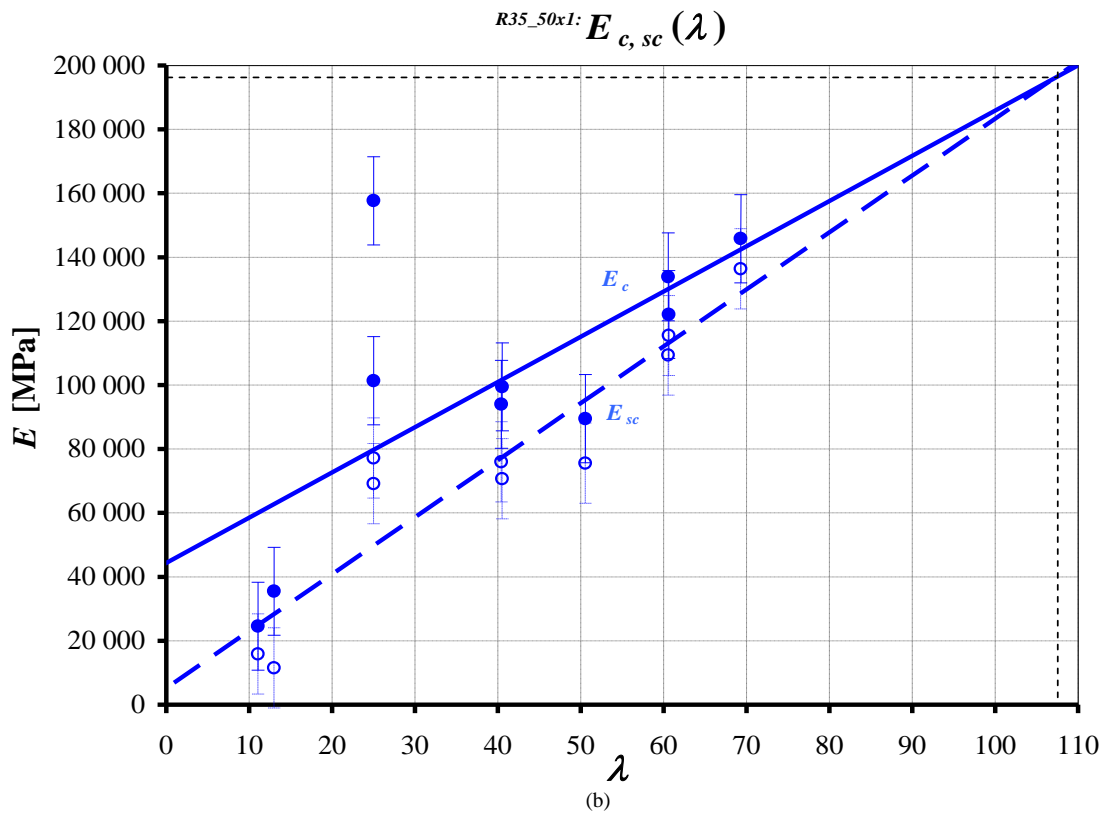
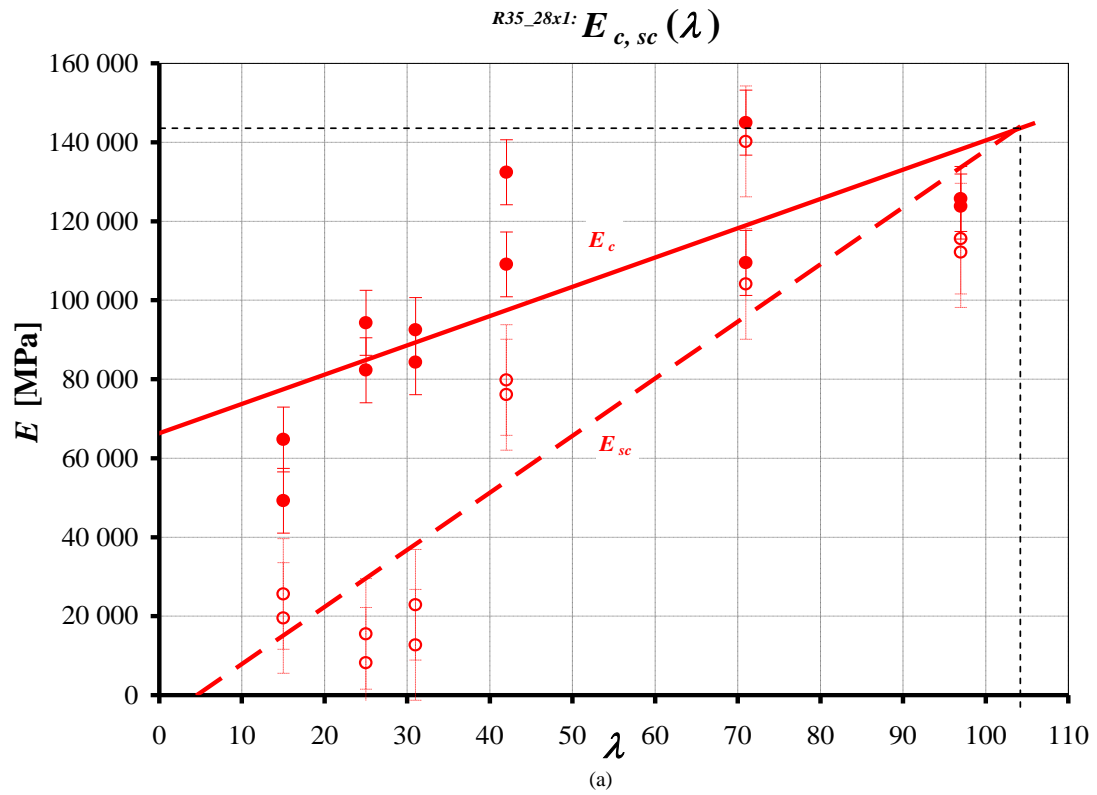


Fig. 25: Approximated functions $E_c(\lambda)$ and $E_{sc}(\lambda)$ of the specimens with cross-section (a) $\phi 26 \times 1$ and (b) $\phi 50 \times 1$ made of steel R35, compressed through ball-and-socket joints with friction

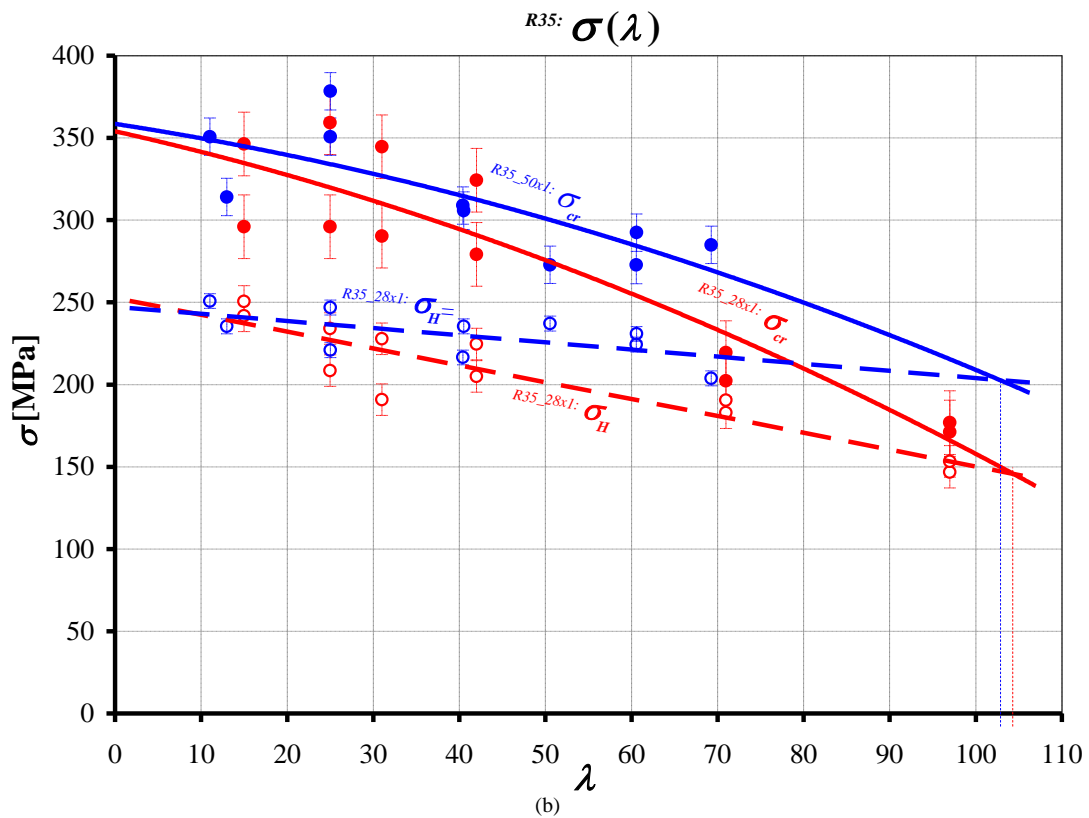
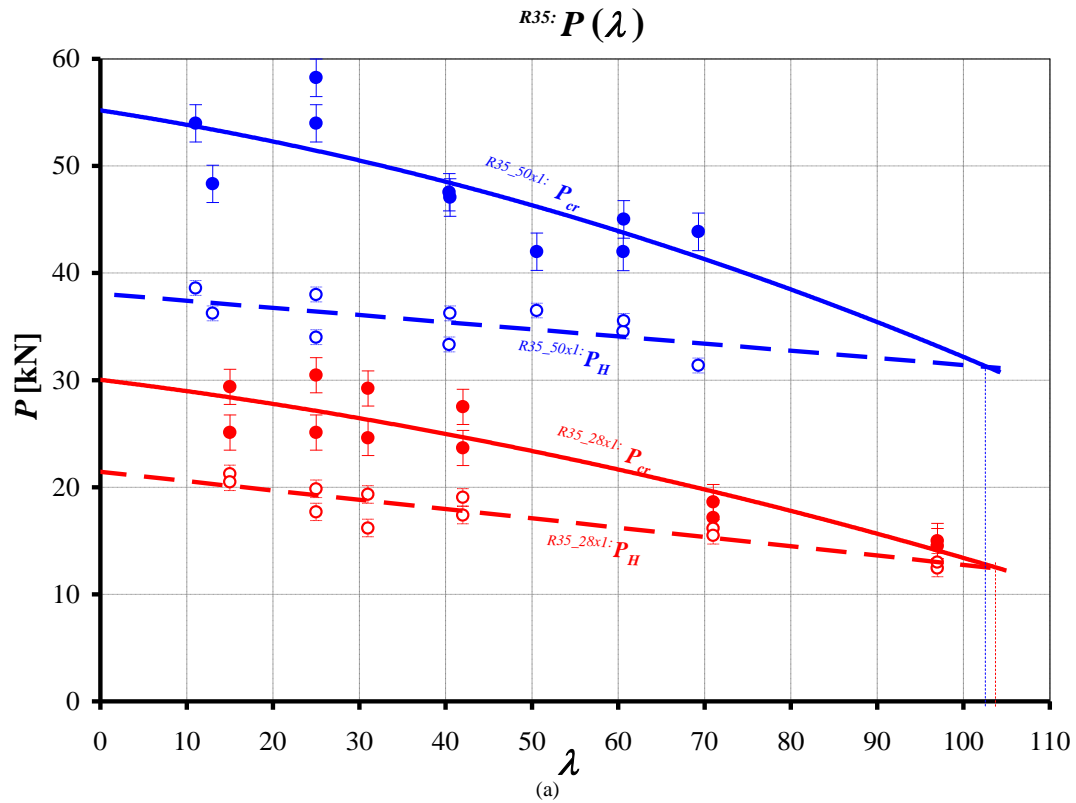


Fig. 26: Sets of the approximated functions: (a) $P(\lambda)$ and (b) $\sigma(\lambda)$ of the specimens made of steel R35, compressed through ball-and-socket joints with friction

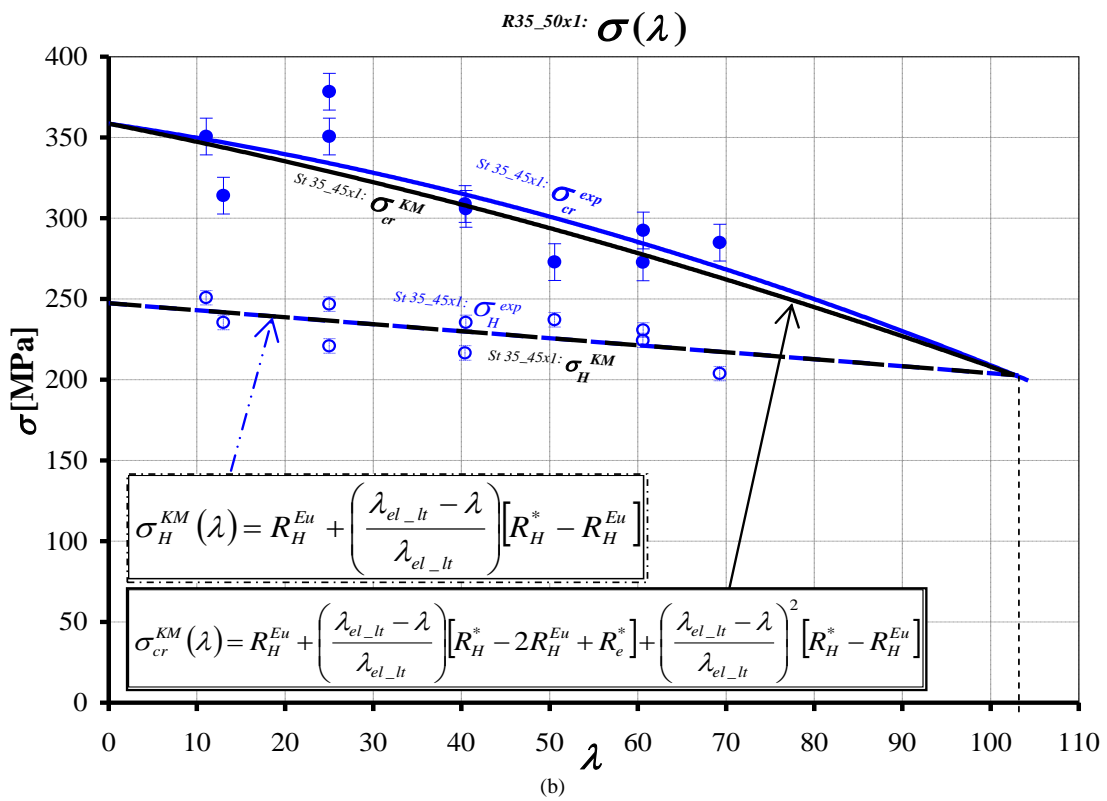
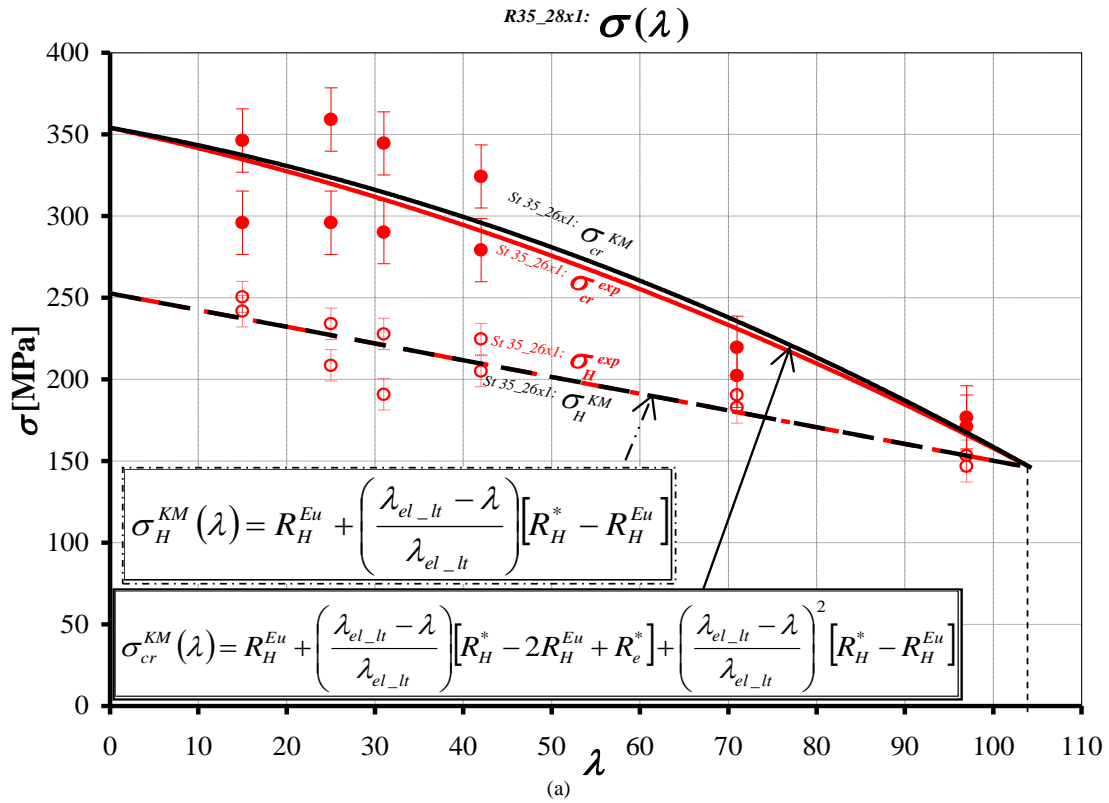


Fig. 27: Sets of the functions: approximated experimental $\sigma^{exp}(\lambda)$ and theoretical $\sigma^{KM}(\lambda)$ of the specimens made of steel R35: (a) $\phi 28 \times 1$ and (b) $\phi 50 \times 1$, compressed through ball-and-socket joints with friction

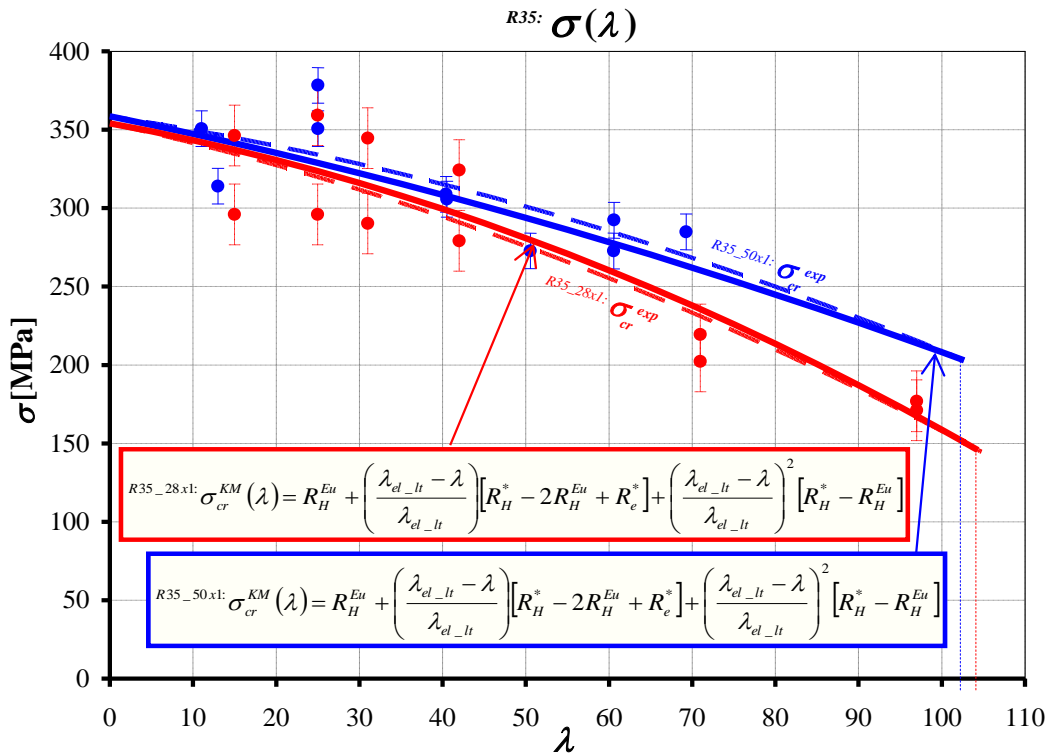


Fig. 28: Functions $\sigma_{cr}(\lambda)$ according to the modified Engesser-Kármán -Shanley's hypothesis and approximated curves obtained from experiments for columns $\phi 50 \times 1$ and $\phi 28 \times 1$ made of steel R35

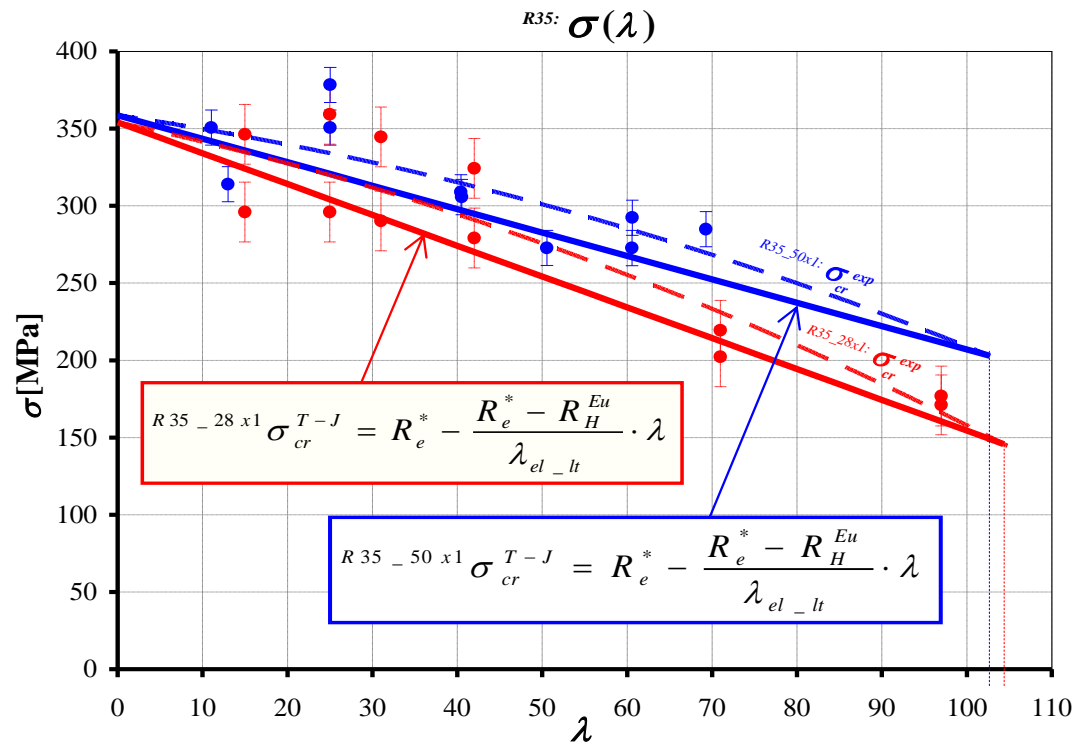


Fig. 29: Functions $\sigma_{cr}(\lambda)$ according to the (Tetmajer, 1886; Jasiński, 1894) simplification and approximated curves obtained from experiments for columns $\phi 50 \times 1$ and $\phi 28 \times 1$ made of steel R35

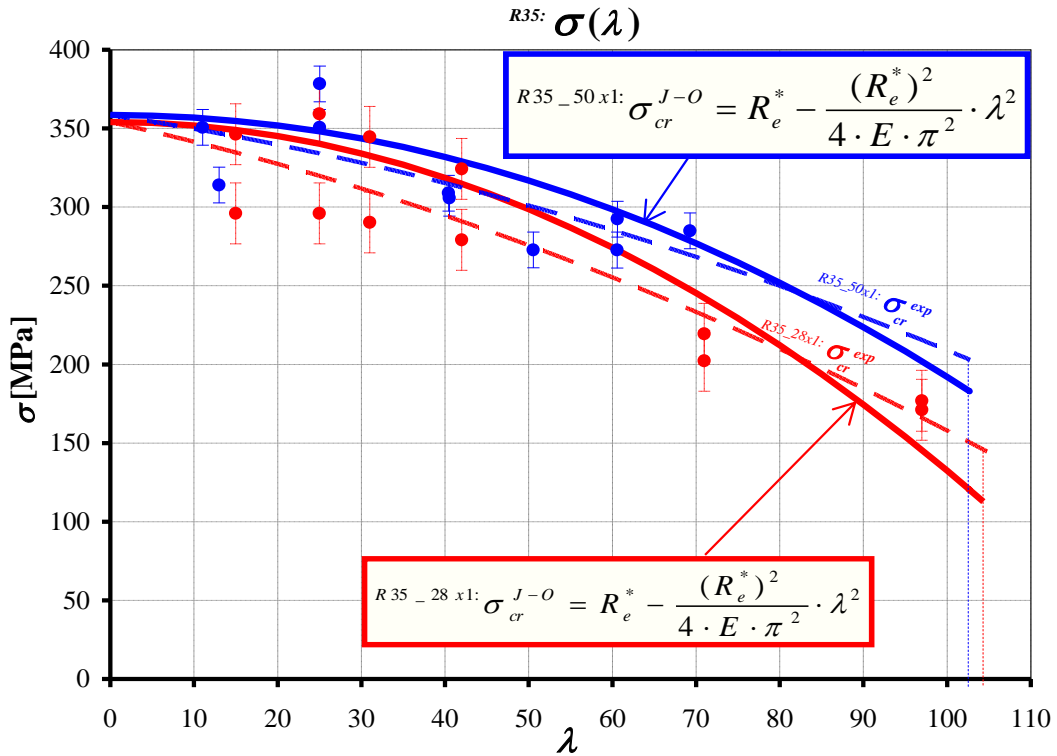


Fig. 30: Functions $\sigma_{cr}(\lambda)$ according to Johnson-Ostenfeld's (1898) simplification and approximated curves obtained from experiments for columns $\phi 50 \times 1$ and $\phi 28 \times 1$ made of steel R35

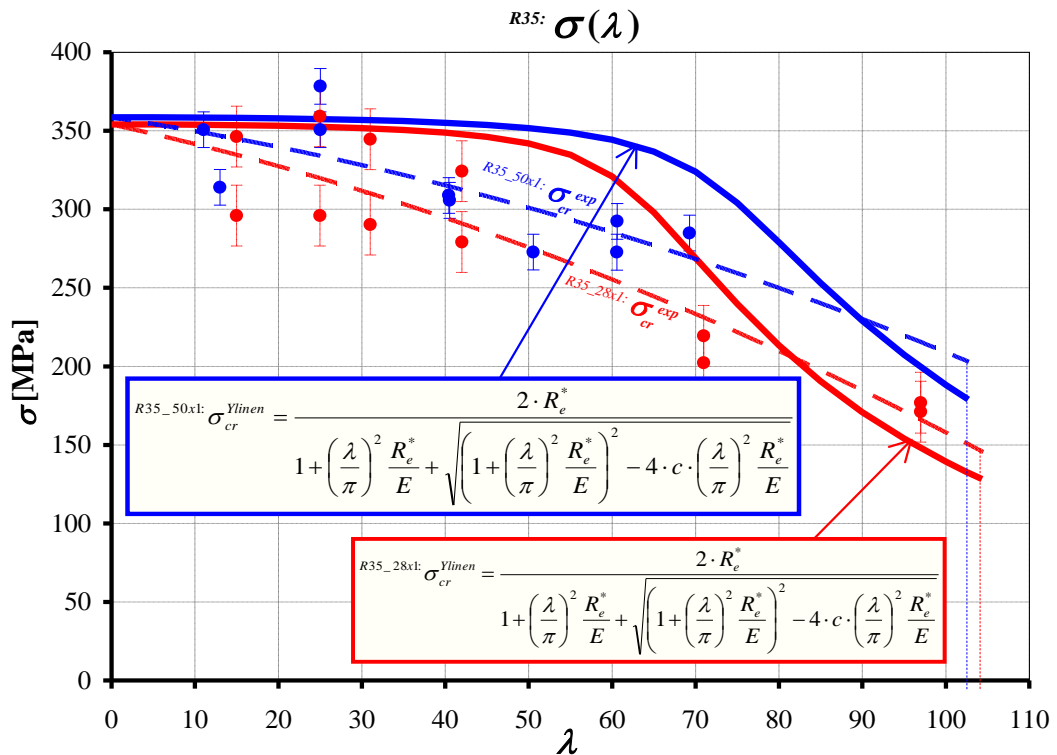


Fig. 31: Functions $\sigma_{cr}(\lambda)$ according to Ylisen's hypothesis (1956) and approximated curves obtained from experiments for columns $\phi 50 \times 1$ and $\phi 28 \times 1$ made of steel R35

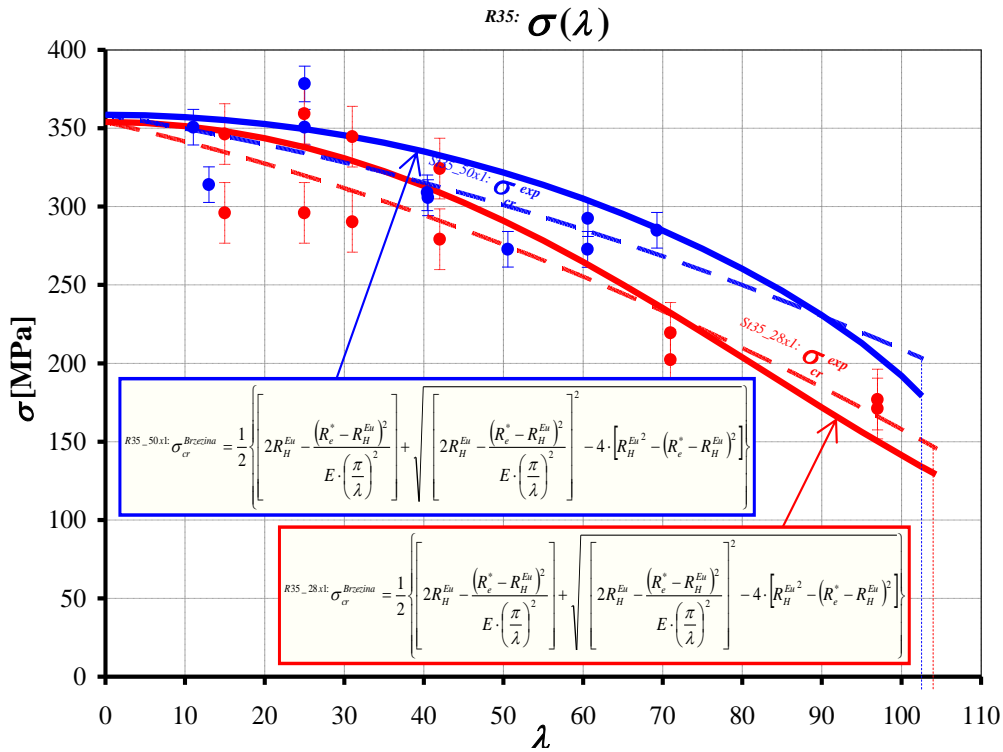


Fig. 32: Functions $\sigma_{cr}(\lambda)$ according to the Březina's hypothesis (1966) and approximated curves obtained from experiments for columns $\phi 50 \times 1$ and $\phi 28 \times 1$ made of steel R35

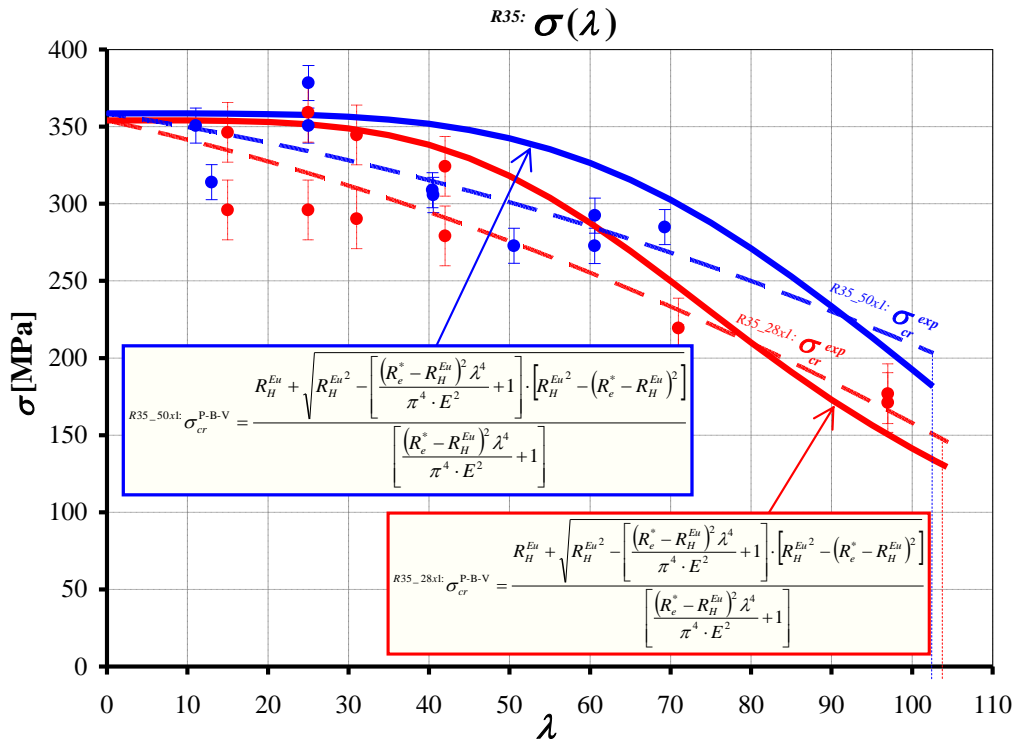


Fig. 33: Functions $\sigma_{cr}(\lambda)$ according to the (Pearson, 1950; Bleich, 1952; Vol'mir's, 1965) hypothesis and approximated curves from experiments for columns $\phi 50 \times 1$ and $\phi 28 \times 1$ made of steel R35

Table 1: Maximal differences Δ [%] between results obtained according to the author's approximated hypothesis and results obtained from of the tests on the columns $\phi 50 \times 1$ made of steel R35

$R35_{50x1}: \Delta_{cr_max}^{exp-KM} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{KM})_{max}$	$R35_{50x1}: \Delta_{cr_max}^{exp/KM} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{KM}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
7,06	2,41
$R35_{50x1}: \Delta_{cr_max}^{exp-(T-J)} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{T-J})_{max}$	$R35_{50x1}: \Delta_{cr_max}^{exp/(T-J)} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{T-J}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
18,17	6,19
$R35_{50x1}: \Delta_{cr_max}^{exp-(J-O)} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{J-O})_{max}$	$R35_{50x1}: \Delta_{cr_max}^{exp/(J-O)} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{J-O}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
-16,70	-5,38
$R35_{50x1}: \Delta_{cr_max}^{exp-Ylinen} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{Ylinen})_{max}$	$R35_{50x1}: \Delta_{cr_max}^{exp/Ylinen} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{Ylinen}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
-59,90	-21,63
$R-35_{50x1}: \Delta_{cr_max}^{exp-Brezina} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{Brezina})_{max}$	$R-35_{50x1}: \Delta_{cr_max}^{exp/Brezina} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{Brezina}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
24,33	11,99
$R35_{50x1}: \Delta_{cr_max}^{exp-(P-B-V)} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{P-B-V})_{max}$	$R35_{50x1}: \Delta_{cr_max}^{exp/(P-B-V)} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{P-B-V}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
-42,06	-14,42

Table 2: Maximal differences Δ [%] between results obtained according to the author's approximated hypothesis and results obtained from of the tests on the columns $\phi 28 \times 1$ made of steel R35

$R35_{28x1}: \Delta_{cr_max}^{exp-KM} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{KM})_{max}$	$R35_{28x1}: \Delta_{cr_max}^{exp/KM} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{KM}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
-5,21	-2,01
$R35_{28x1}: \Delta_{cr_max}^{exp-(T-J)} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{T-J})_{max}$	$R35_{28x1}: \Delta_{cr_max}^{exp/(T-J)} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{T-J}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
21,46	8,26
$R35_{28x1}: \Delta_{cr_max}^{exp-(J-O)} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{J-O})_{max}$	$R35_{28x1}: \Delta_{cr_max}^{exp/(J-O)} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{J-O}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
33,06	22,67
$R35_{28x1}: \Delta_{cr_max}^{exp-Ylinen} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{Ylinen})_{max}$	$R35_{28x1}: \Delta_{cr_max}^{exp/Ylinen} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{Ylinen}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
-68,93	-25,94
$R35_{28x1}: \Delta_{cr_max}^{exp-Brezina} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{Brezina})_{max}$	$R35_{28x1}: \Delta_{cr_max}^{exp/Brezina} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{Brezina}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
16,82	11,51
$R35_{28x1}: \Delta_{cr_max}^{exp-(P-B-V)} [\text{MPa}] = (\sigma_{cr}^{exp} - \sigma_{cr}^{P-B-V})_{max}$	$R35_{28x1}: \Delta_{cr_max}^{exp/(P-B-V)} [\%] = \left(\frac{\sigma_{cr}^{exp} - \sigma_{cr}^{P-B-V}}{\sigma_{cr}^{exp}} \right)_{max} \cdot 100\%$
-44,09	-15,45

For the columns $\phi 28 \times 1$ on the base of the tests were assumed: $E^* = 143\,230$ MPa, $\lambda_{el-lt} = 104.331$, $R_e^* = 354.05$ MPa, $R_H^* = 252.68$ MPa, $R_H^{Eu} = 145.803$ MPa.

The average values were assumed: $E^* = 166\,614$ MPa, $\lambda_{el-lt} = 102.6$, $R_e^* = 346.54$ MPa, $R_H^* = 268.24$ MPa, $R_H^{Eu} = 156.0$ MPa.

The tests were executed by ball-and-sockets joints what caused increasing partition of friction between balls and sockets with increasing load.

Conclusion

For Johnson-Ostenfeld's, Ylinen's, Březina's and Pearson-Bleich-Vol'mir's theories the surface functions are almost flat in the most part-the bronze parts of the surface functions in Figs. 3 to 10.

The formulas given by those theories don't allow presenting them depending on the transverse cross-section area A and slenderness ratio λ together - for what allows the technical stability theory for columns in elastic-plastic states. That proves that those theories are simplified and limited.

They don't allow presenting the functions for the shell stresses and strains-for what allows the technical stability theory for columns in elastic-plastic states.

Besides, it was concluded that:

- For increasing value of a slenderness ratio λ the values of the compress modulus E_c and secant compress modulus E_{sc} were also increasing: $E_c(\lambda_i) < E_c(\lambda_{i+1}) < E_c(\lambda_{i+2})$ and $E_{sc}(\lambda_i) < E_{sc}(\lambda_{i+1}) < E_{sc}(\lambda_{i+2})$ - Fig. 1 to 25
- For the slenderness ratio limiting elastic states λ_{el-lt} the compress modulus E_c and secant compress modulus E_{sc} were equal and that value was signed as $E^* = E_c(\lambda_{el-lt}) = E_{sc}(\lambda_{el-lt}) \approx E$ - Figs. 1 to 25
- With increasing of the transverse cross-section area A the shortness $\Delta L_H(\lambda)$ is increasing too, but $\Delta L_{cr}(\lambda)$ is decreasing and they meet together at λ_{el-lt} - Fig. 21a and 23a
- The strains $\varepsilon(\lambda)$ are decreasing exponentially-Fig. 21b and 23b
- With increasing of the transverse cross-section area A the forces P_{cr}^* and P_H^* are increasing too, but the stresses R_e^* and R_H^* are almost constant-Fig. 26
- With increasing of the transverse cross-section area A the stresses $\sigma_{cr}(\lambda)$ and $R_H(\lambda)$ are increasing too and the stress $R_H^{E_{tu}}$ is also increasing-Fig. 26. The stress $R_H(\lambda)$ is changing because its values were measured from the curves $\sigma(\varepsilon)$ obtained from the experiments and they consisted the change of an elastic part in the critical transverse cross-sections A with the changing slenderness ratio
- With increasing of transverse cross-section area A the slenderness ratios limiting the elastic states λ_{el-lt} are almost the same-Fig. 26

It was also concluded that the approximated theory of technical stability for columns in the elastic-plastic states gives the possibility to determine the stress $R_H(\lambda)$ as well as $\sigma_{cr}(\lambda)$.

Ethics

The content of this study is the changed preprint (Murawski, 2020b) according to the issuer requirements and reviewers' and own author's corrections.

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